



The 1st CEPC International Accelerator Review Committee Meeting in 2022
from Tuesday, June 7, 2022 at 15:00 to Friday, June 17, 2022 at 19:10 (Asia/Shanghai)

Cryogenic System for CEPC

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Miaofu Xu, **Zhengze Chang**

On behalf of the CEPC cryogenic group

June, 9th, 2022

- **Introduction**
- CEPC SRF cryogenic system
- CEPC SC magnets cryogenic system
- Key technology
 - **2K JT heat exchanger R&D and test**
- Summary

Booster ring:

- 1.3 GHz 9-cell cavities, 96 cavities
- 12 cryomodules
- 3 cryomodules/each station
- Temperature: 2K

Collider ring:

- 650MHz 2-cell cavities, 240 cavities
- 40 cryomodules
- 10 cryomodules/each station
- Temperature: 2K

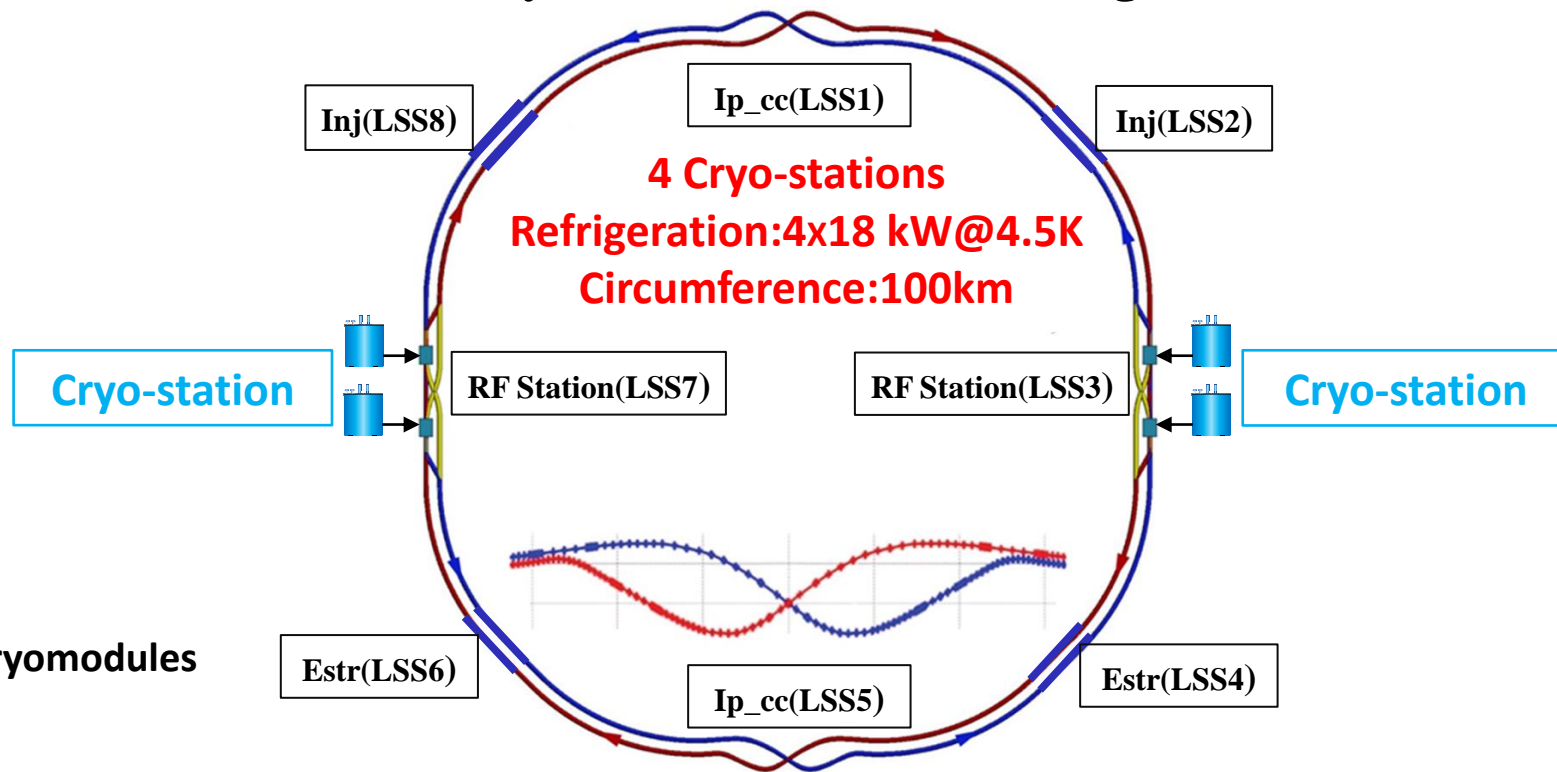
IR magnets:

- 4 IR magnets, 32 Sextupole magnets, 36 cryomodules
- 18 cryomodules/each station
- Temperature: 4.5K

Detectors:

- 2 detector magnets
- Thermosiphon cooling

Layout of CEPC Double Ring



- Introduction and Layout
- **CEPC SRF cryogenic system**
- CEPC SC magnets cryogenic system
- Key technology
- Summary



Estimated heat loads for SC cavities-Modif. Scheme



Higgs Mode 50MW	Unit	Collider			Booster		
		40-80K	5-8K	2K	40-80K	5-8K	2K
Predicted static heat load per cryomodule	W	300	60	12	140	20	3
Cavity dynamic heat load per cryomodule	W	0	0	153.59	0	0	13.98
HOM dynamic heat load per cryomodule	W	20	12	2	2	1	1
Input coupler dynamic heat load per cryomodule	W	60	40	6	40	3	0.4
Module dynamic heat load	W	80	52	161.59	42	4	15.38
Each Module total heat load	W	380	112	173.59	182	24	18.38
Cryomodule number	-	56	56	56	12	12	12
EVB heat loss	W	50	10	10	50	10	10
EVB number	-	40	40	40	12	12	12
MDVB heat loss	W	50	30	30	50	30	5
MDVB number	-	4	4	4	4	4	4
Total cryogenic transfer line length	m	728	728	728	136	136	136
cryogenic transfer line heat loss per meter	W/m	2	0.5	0.3	2	0.5	0.3
Total cryogenic transfer line heat loss	W	1456	364	218.4	272	68	40.8
Total heat load	kW	24.936	7.156	10.45944	3.256	0.596	0.40136
Total predicted mass flow	g/s	82.42	152.26	373.72	13.34	12.73	18.324
Overall net cryogenic capacity multiplier		1.54	1.54	1.54	1.54	1.54	1.54
4.5K equiv. heat load with multiplier	kW	2.89	9.97	51.56	0.37	0.83	1.98
4.5K equiv. heat load with multiplier	kW		64.41			3.19	
Total 4.5K equiv. heat load with multiplier	kW		4 × 16.85 kW		67.60		

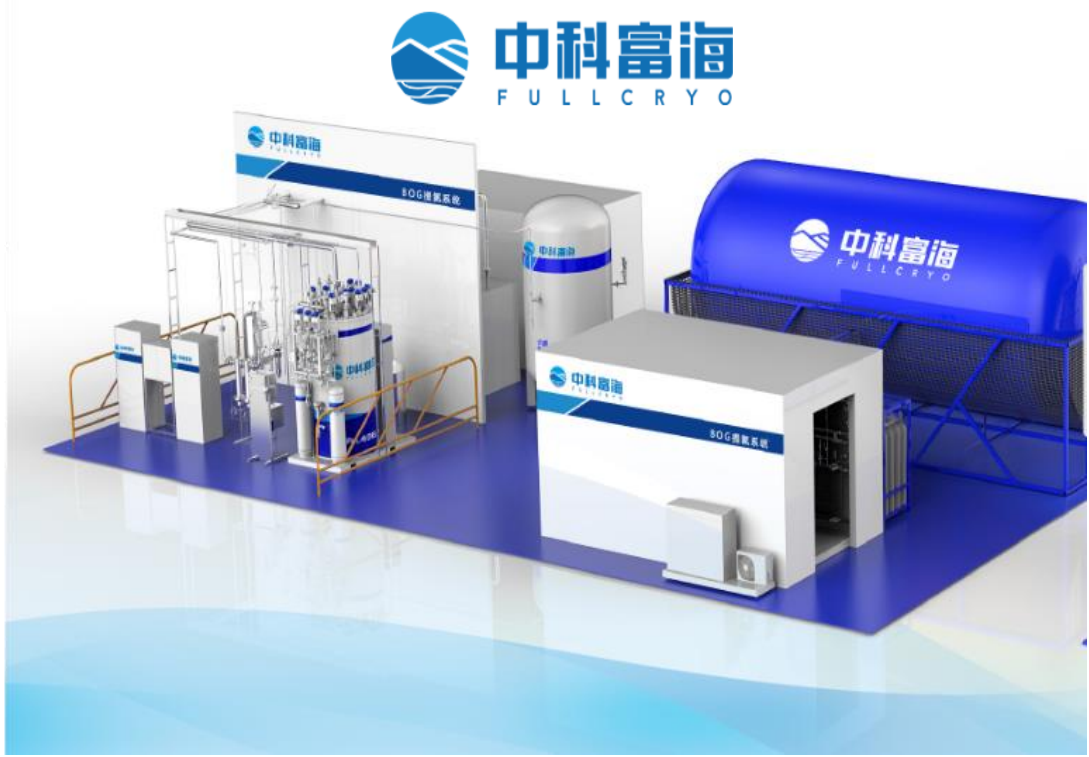
•Four individual refrigerators (4 × 18kW@4.5K) will be employed for the CEPC cavity cryogenic system.



Large Helium Refrigerator Market Research



- ◆ Domestic refrigerator / liquefier
- Institute of physics and Chemistry, CAS (FULL CRYO) has: 2kW@20K, 10kW@20K, etc.; 2.5KW@4.5K or 500W@2K Has been successfully developed;
- **10kW@4.5K or 18kW@4.5K (CEPC) has been planned with the foundation of the pilot project of Chinese Academy of Science.**
- Institute of Plasma Physics, CAS: existing 500W@4.5K or 150L / h, 2.2kW@4.5K , planned 5kW@4.5K



【中国科学报】我国研发成功-271℃超流氦大型低温制冷装备

中科院理化所 4月17日

2.5KW@4.5K or 500W@2K 百瓦级



4月15日，由财政部支持、中国科学院理化技术研究所承担的国家重大科研装备研制项目“液氦到超流氦温区大型低温制冷系统研制”通过验收及成果鉴定，标志着我国打破了发达国家的技術垄断，具备了研制液氦温度（零下269摄氏度）千瓦级和超流氦温区（零下271摄氏度）百瓦级大型低温制冷装备的能力，将可满足大科学工程、航天工程、氦资源开发等国家战略高技术发展的迫切需要。

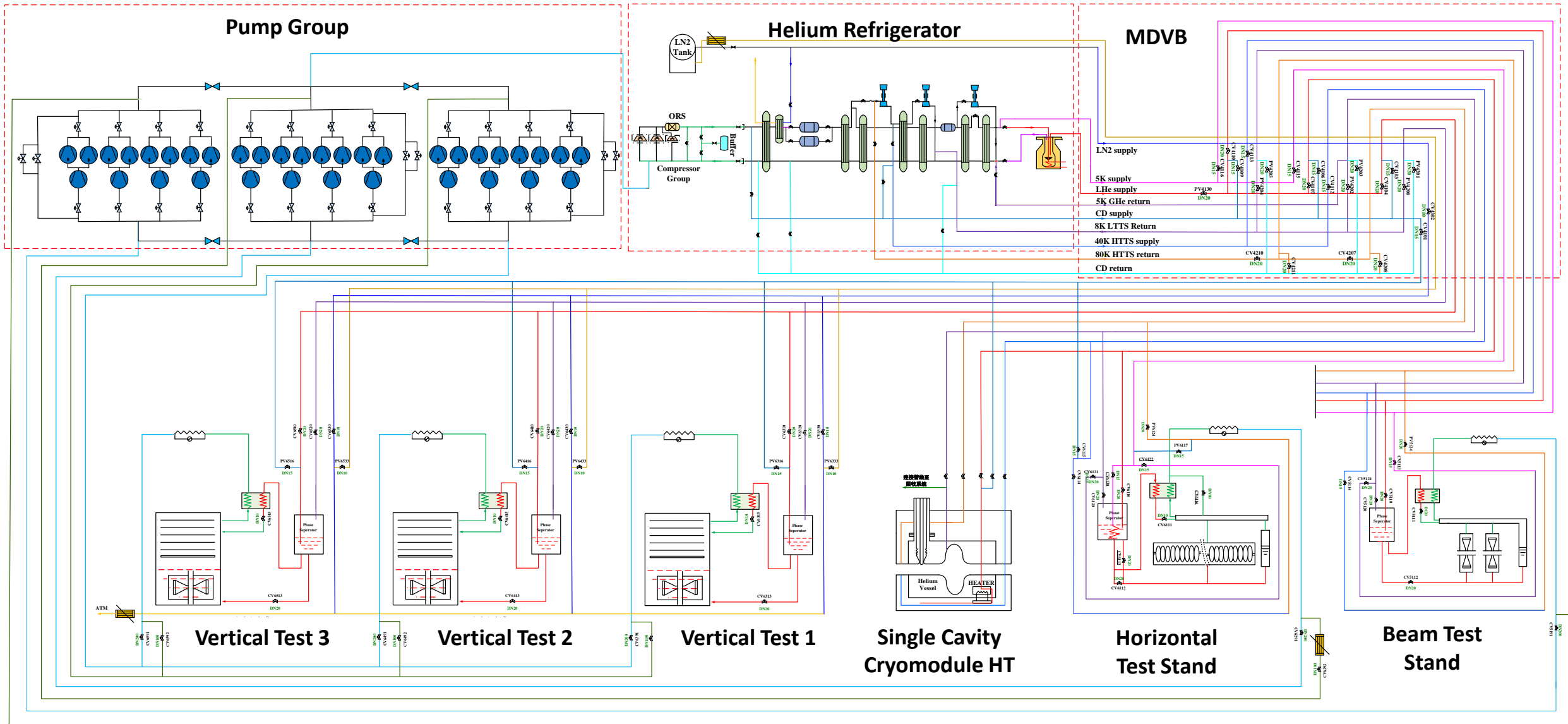
液氦温度低至-269℃，是深低温区最重要的低温源。液氦到超流氦温区大型低温制冷装备是航空航天、氢能源、国家大科学装置等战略领域不可或缺的核心基础平台。但是，越往低温推进，制冷技术难度、成本、功耗都呈几何级数上升。核心技术的缺失，使得我国大型低温制冷装备全部依赖进口，关键核心部件以及用于特殊领域的专用制冷装备国外对我禁运。

中科院理化所具有几十年低温技术的深厚积累，在洪朝生院士、周远院士带领下，通过一代代低温科技工作者的不懈努力，坚持走自主创新道路，经过五年多的拼搏奋斗，在温度（零下253度）制冷机的基础上，自主研发制出了技术指标为2500W@4.5K和2.5KW@2K的大型氦制冷机。

百瓦级 零下271℃
我国大型低温制冷技术取得重大突破

国家重大科研装备研制项目“液氦到超流氦温区大型低温制冷系统研制”通过验收及成果鉴定

是航空航天、氢能源储运、氦资源开发等领域及一批大科学装置不可或缺的基础装备



cryogenic hall



RF hall



Tanks zone



VT stands

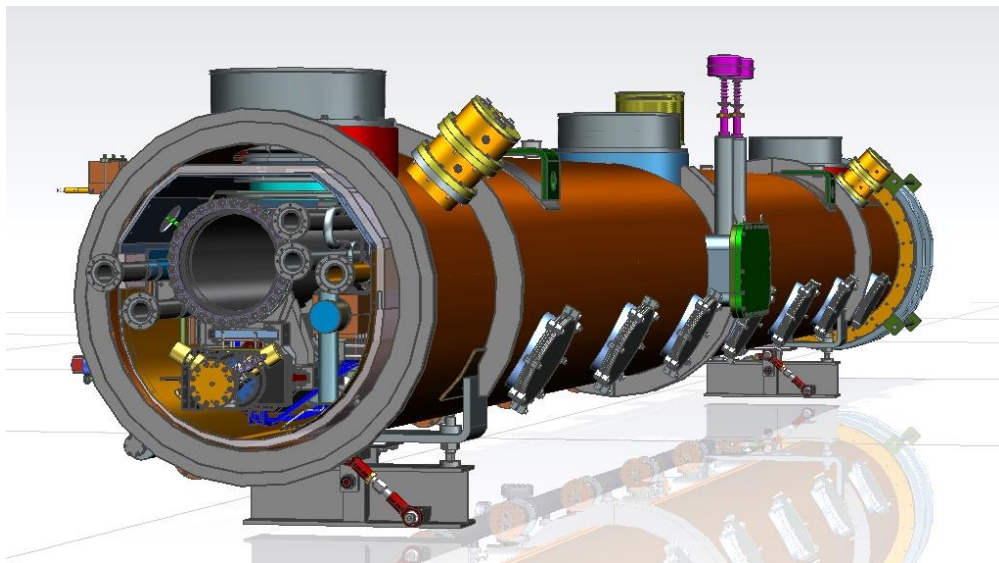


HT test stand



Beam test stand





Design Goals:

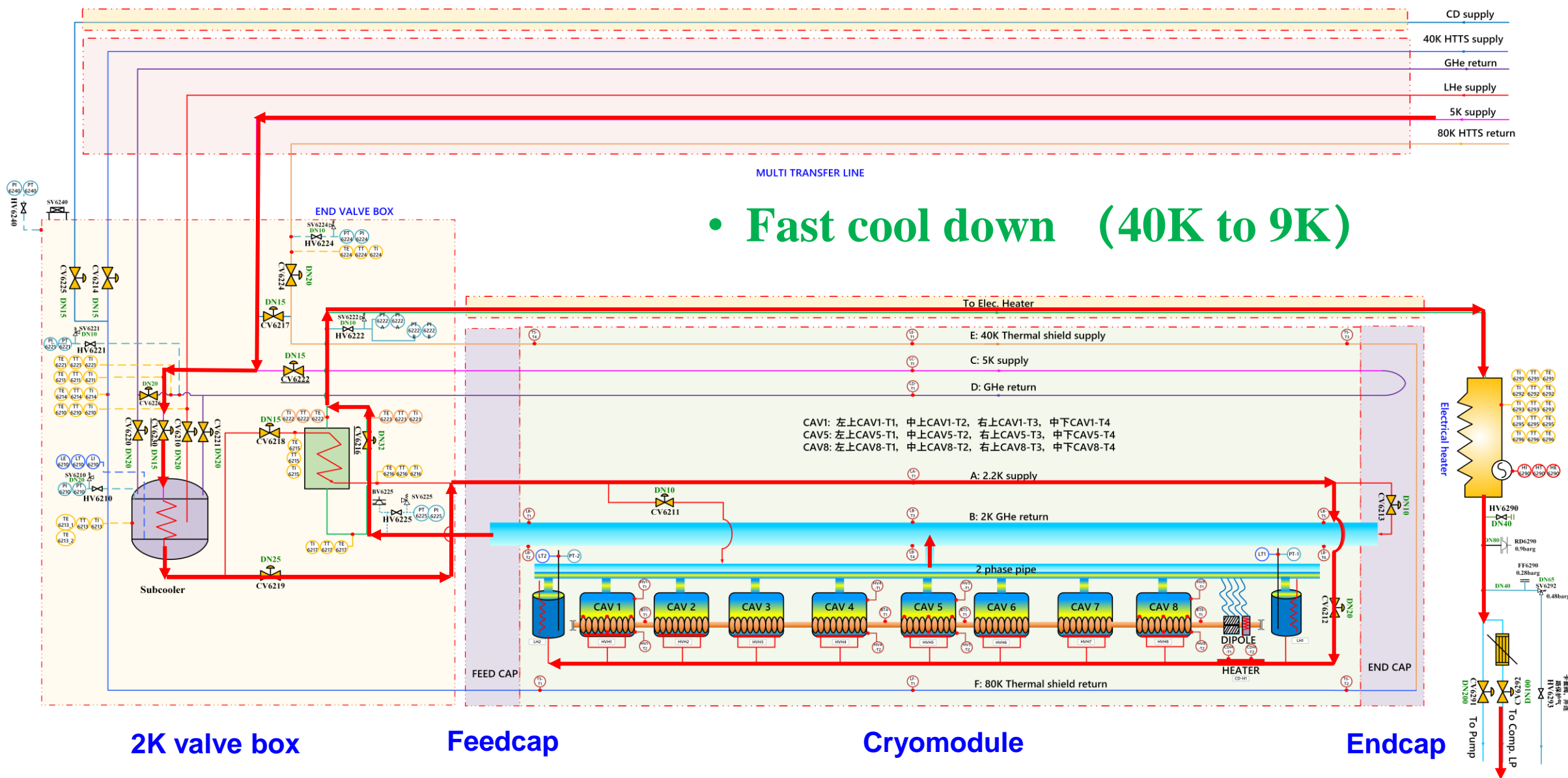
- Low heat loss
- Fast cool down

XFEL / LCLS-II type Cryomodule for High Q Cavity

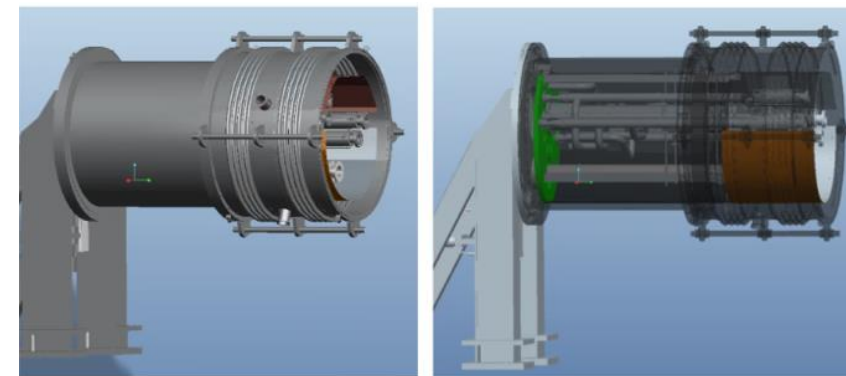
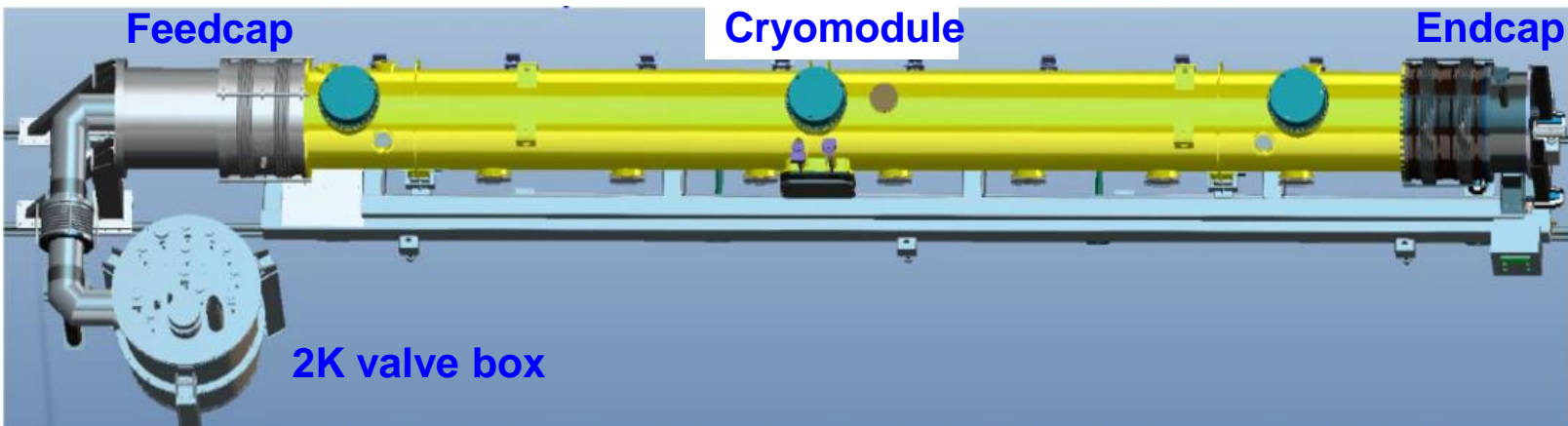
- Cryogenic Group in IHEP has manufactured 58 1.3GHz 9-cell Cryomodules for EXFEL cooperated with local companies.
- It's a good foundation for the optimization design for the CEPC cryomodules.



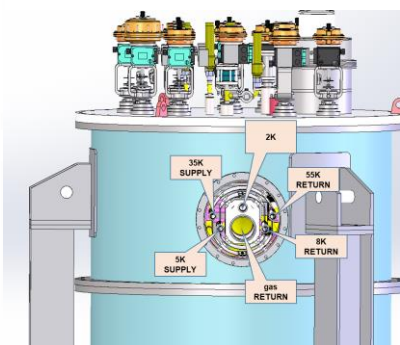
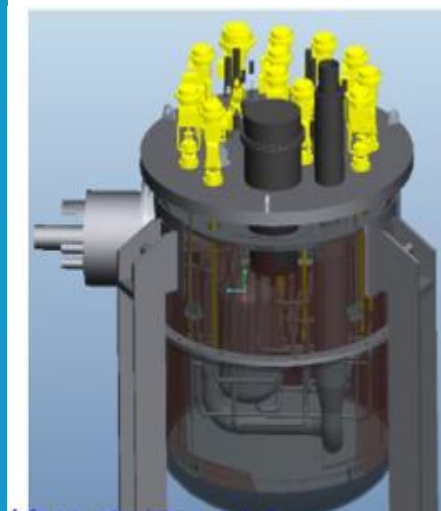
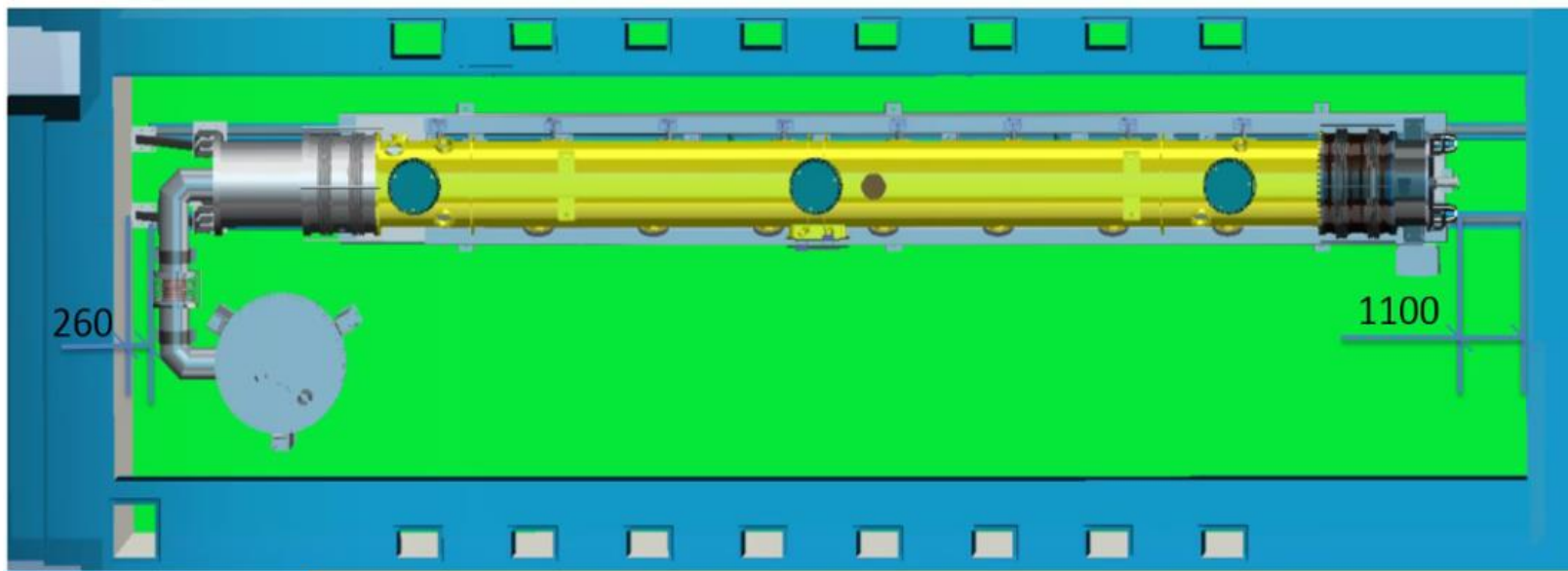
1.3GHz CRYO MODULE TEST STAND



2K horizontal test will be done in PAPS in the last-half 2022 year

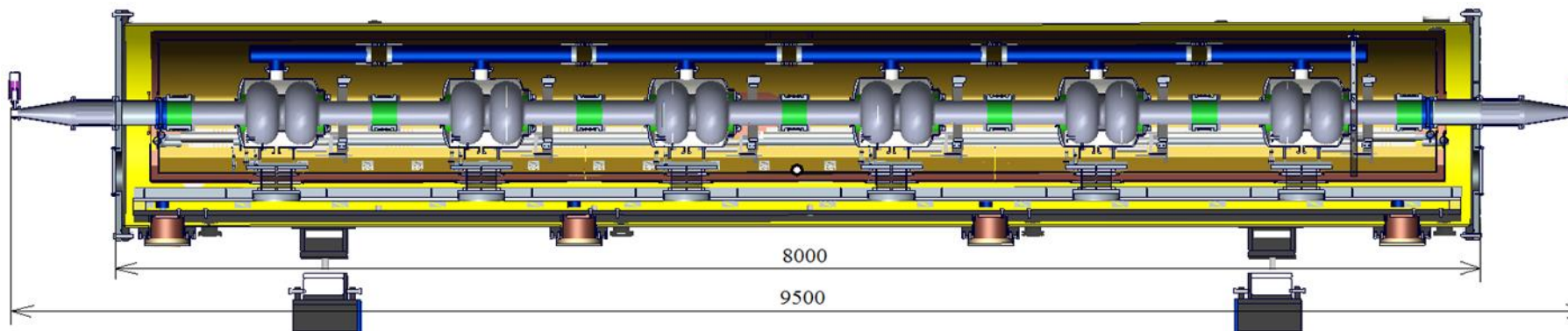


Feedcap



2K valve box

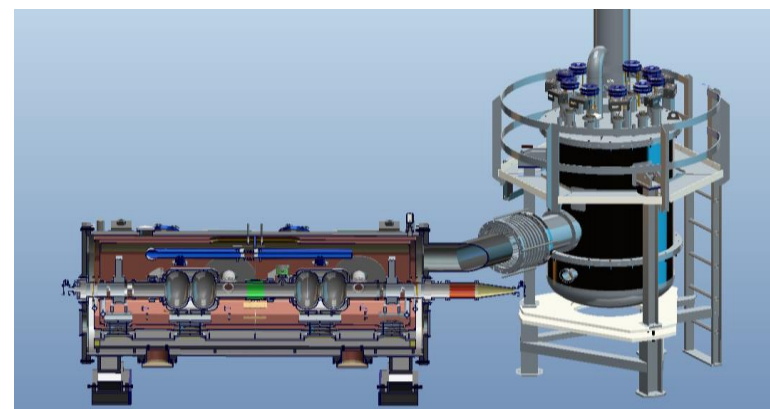
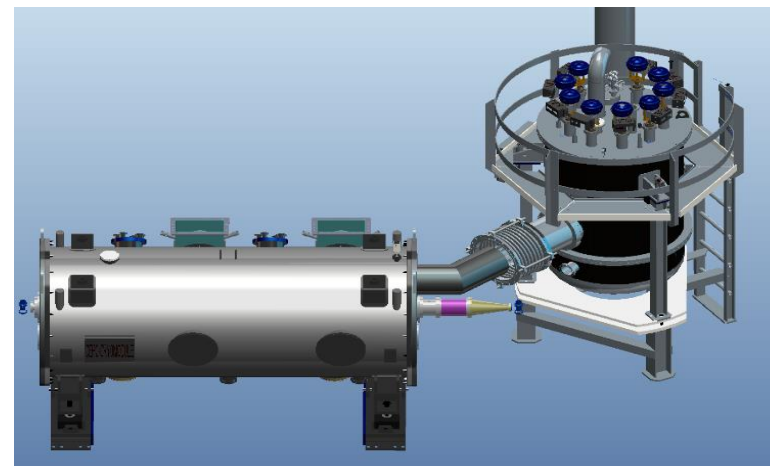
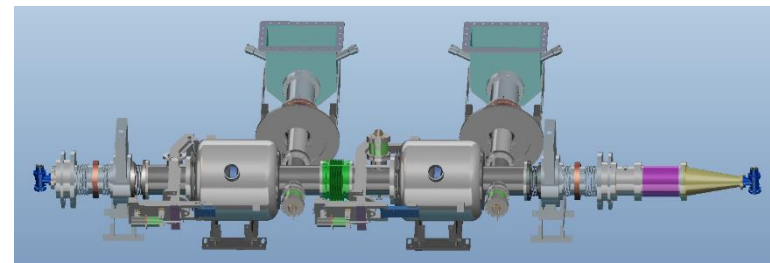
- Including six 2-cell 650 MHz superconducting cavities, six high power couplers, six mechanical tuners and two HOM absorbers



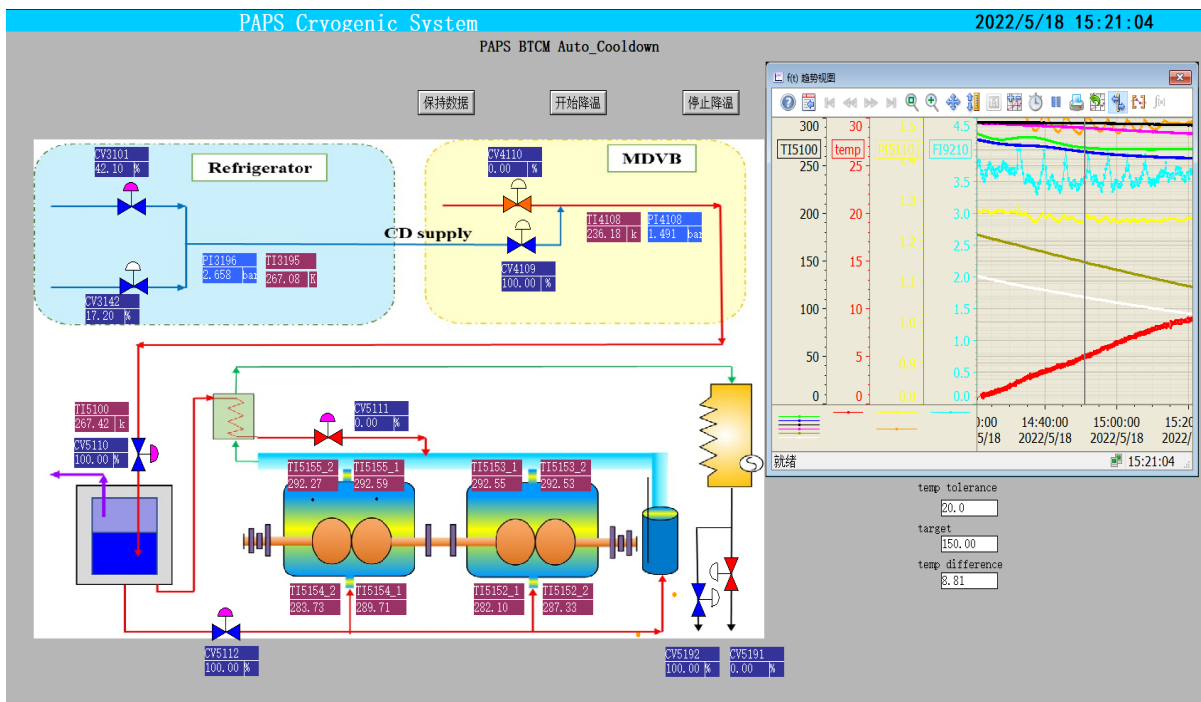
Six 2-cell Cavities Cryomodule	
Overall length(flange to flange, m)	8.0
Diameter of Vacuum vessel ,m	1.3
Beamline height from floor, m	1.5
Cryo-system working temperature, K	2
Number of 200-POST	6

Cryomodule performance		Specification
Number of leakage	He →insulation	0
	He →beam transfer line	0
	Insulation →coupler	0
	Insulation →beam transfer line	0
	Coupler →beam transfer line	0
Alignment x/y inside (Cavities)		±0.5mm
Alignment z inside		within 2 mm
Coupler antenna design z		within 2 mm

- A beam test cryomodule with two 2-cell 650 MHz superconducting cavities has been operated in the PAPS system beam test stand last year.

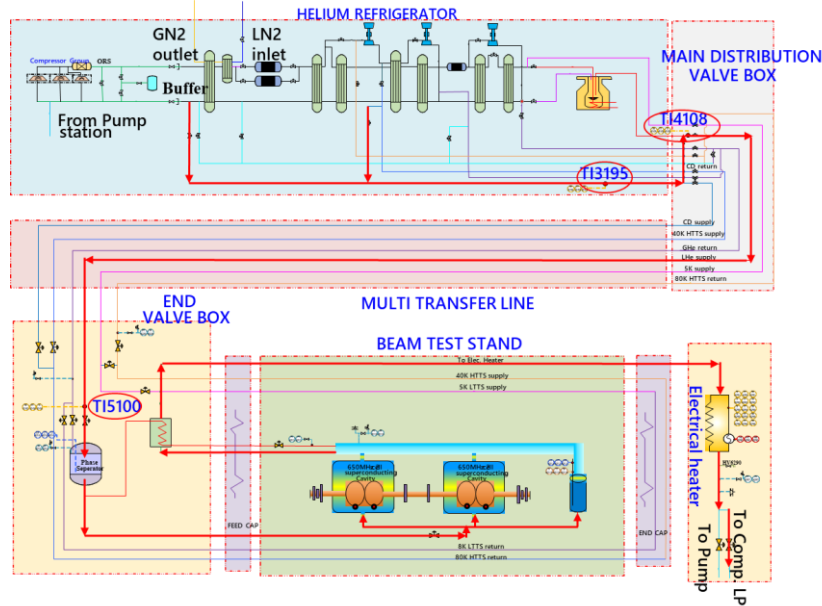


- ❑ Module automatic cooldown research progress: the MPC control method has successfully tested in PAPS BTCM cryostat (**first time in China**)
- ❑ Designed and implemented automatic cool-down/warming-up procedure for SRF cavity cryomodule with Model Predict Control (MPC) method & Artificial Neural Network (ANN)

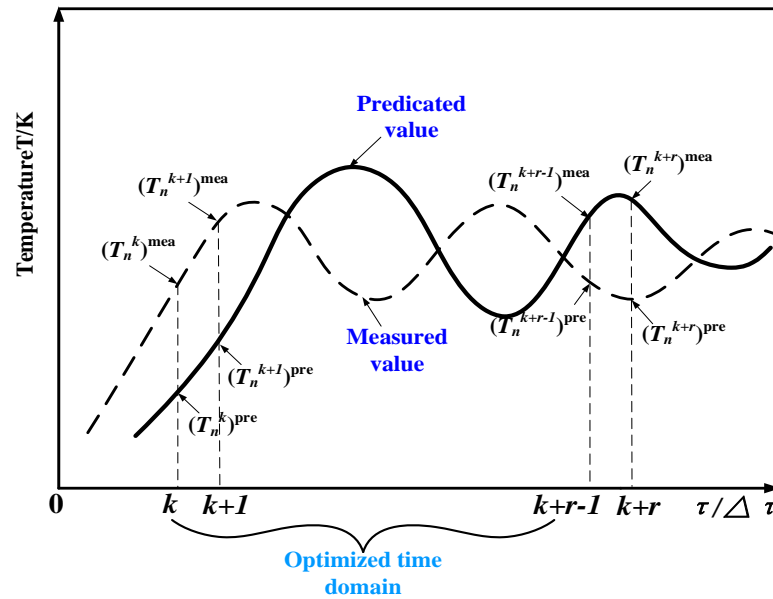


1. 300 to 150 K: < 10 K/hr. Cavity top and bottom $\Delta T < 20$ K
2. 150 to 4.5 K: Cavity surface > 1 K/min
3. 4.5 to 2 K

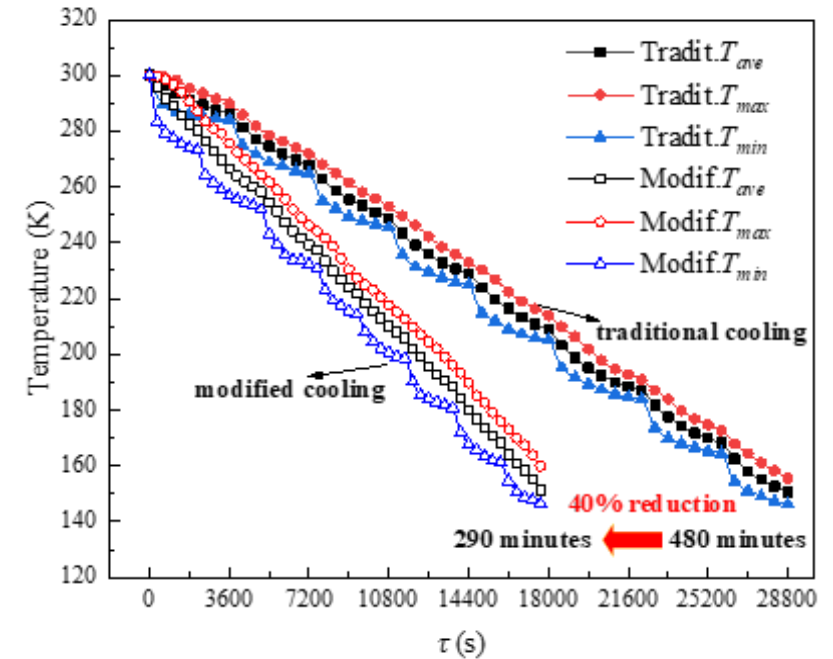
- It is verified that the **MPC method has better control stability.**
- The foundation for a more intelligent automated control of future CEPC large cryogenic systems has been laid.



PFD of beam test stand



Principle diagram of the MPC method



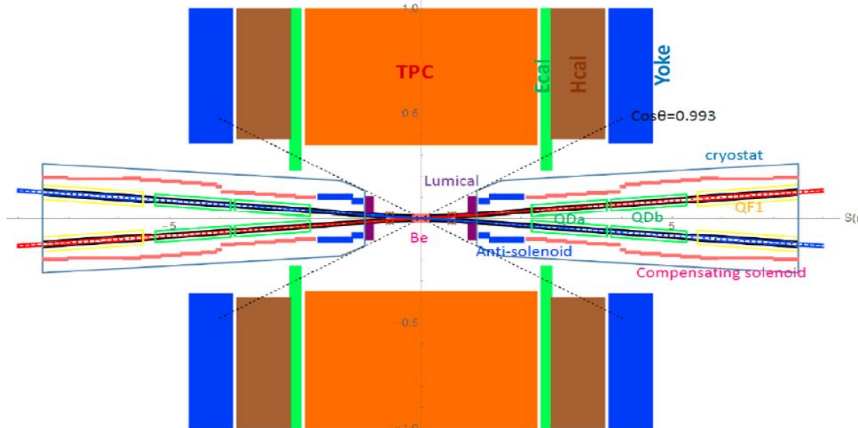
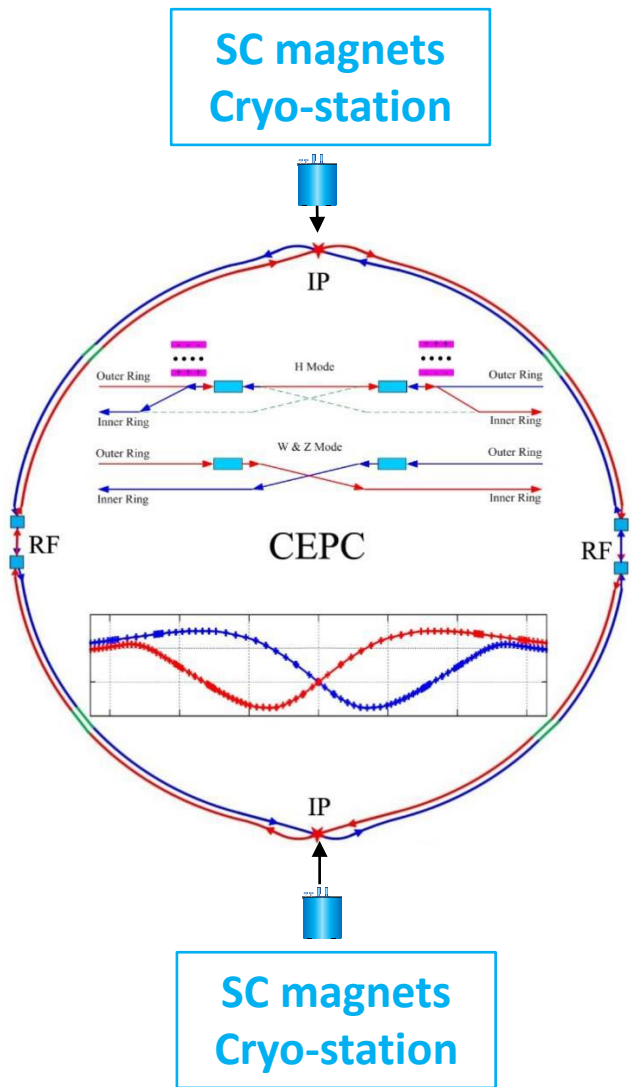
MPC method efficiently decreasing the time of CD

[1] Zheng-ze Chang*, Mei Li,Rui Ge*, Model predictive control of long Transfer-line cooling process based on Back-Propagation neural network, Applied Thermal Engineering, Volume 207,2022,118178,ISSN 1359-4311.

[2] Li M, Chang Z, Zhu K, et al. Unsteady numerical simulation and optimization of 499.8 MHz superconducting cavity cooling process at the High Energy Photon Source (HEPS)[J]. Thermal Science and Engineering Progress, 2021, 26: 101100

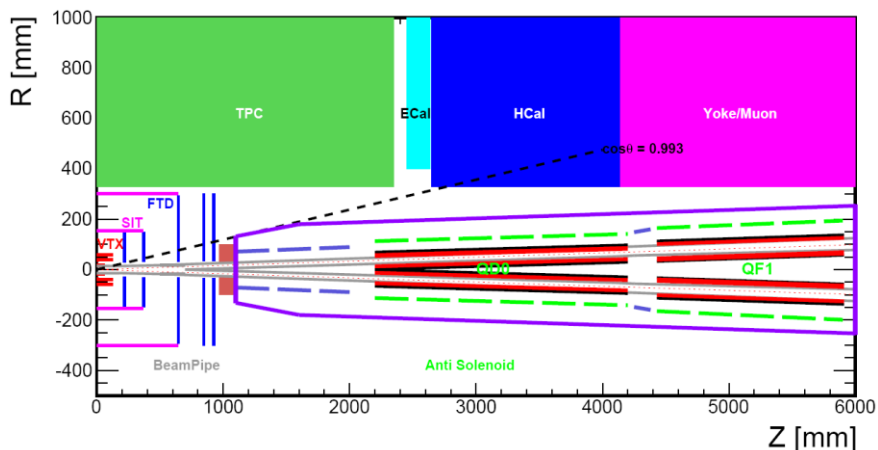
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the detectors are installed at two interaction points (IPs)

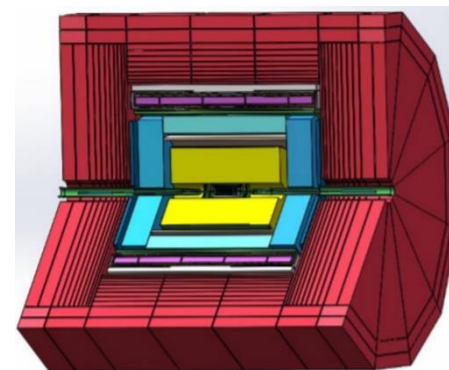


Schematic layout of QD0, QF1, and anti-solenoid

IR SC magnet (QD0, QF1, and anti-solenoid)



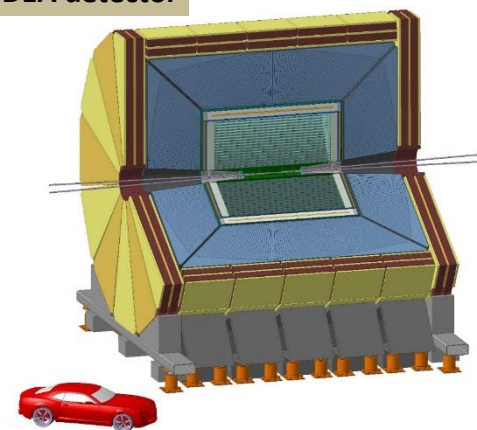
From Zhu Ying-shun



LTS Solenoid :

- Solenoid located outside calorimeter
- Inner diameter 7.2 m, length 7.4 m
- Central field: 3 T
- Superconductor: NbTi
- Operation temperature: 4.2 K

IDEA detector



HTS Solenoid :

- Solenoid located inside calorimeter/less material
- Inner diameter 4 m, length 6 m
- Central field: 2 T
- Superconductor: YBCO
- Operation temperature: 20 K

Zhu Zian, Wang Meifen



Estimated heat loads for SC IR magnets



Name	Unit	4.5K main loop			Thermal Shielding 40~80K		
		No.	Heat load for each	Heat load	No.	Heat load for each	Heat load
IR SC sextupole magnet	W	32	10	320			
Valve Box of IR SC sextupole magnet	W	32	20	640			
Current lead heat load of IR SC sextupole magnet	W	32	—	—			
IR SC magnet	W	4	30	120			
Valve Box of IR SC magnet	W	4	30	120			
Current lead of IR SC magnet	g/s	4	0.5	2			
Main distribution valve box	W	4	50	100			
Cryogenic transfer-line	W	4000	0.3	1200	3424	1.5	5136
Total heat load @4.5K	W			2500			5136
Total heat load @4.5K	W			2500W+2g/s			5136
Total heat load @4.5K	W			2523.45			5136
Coefficient				1			0.05
Total heat load @4.5K	W			2523.45			256.8
Total equiv. heat load @4.5K	W	2780.25					
Total equiv. heat load @4.5K with multiplier 1.5	W	4170					
Cooling capacity of refrigerator@4.5K	W	2	4kW@4.5K	8kW@4.5K			

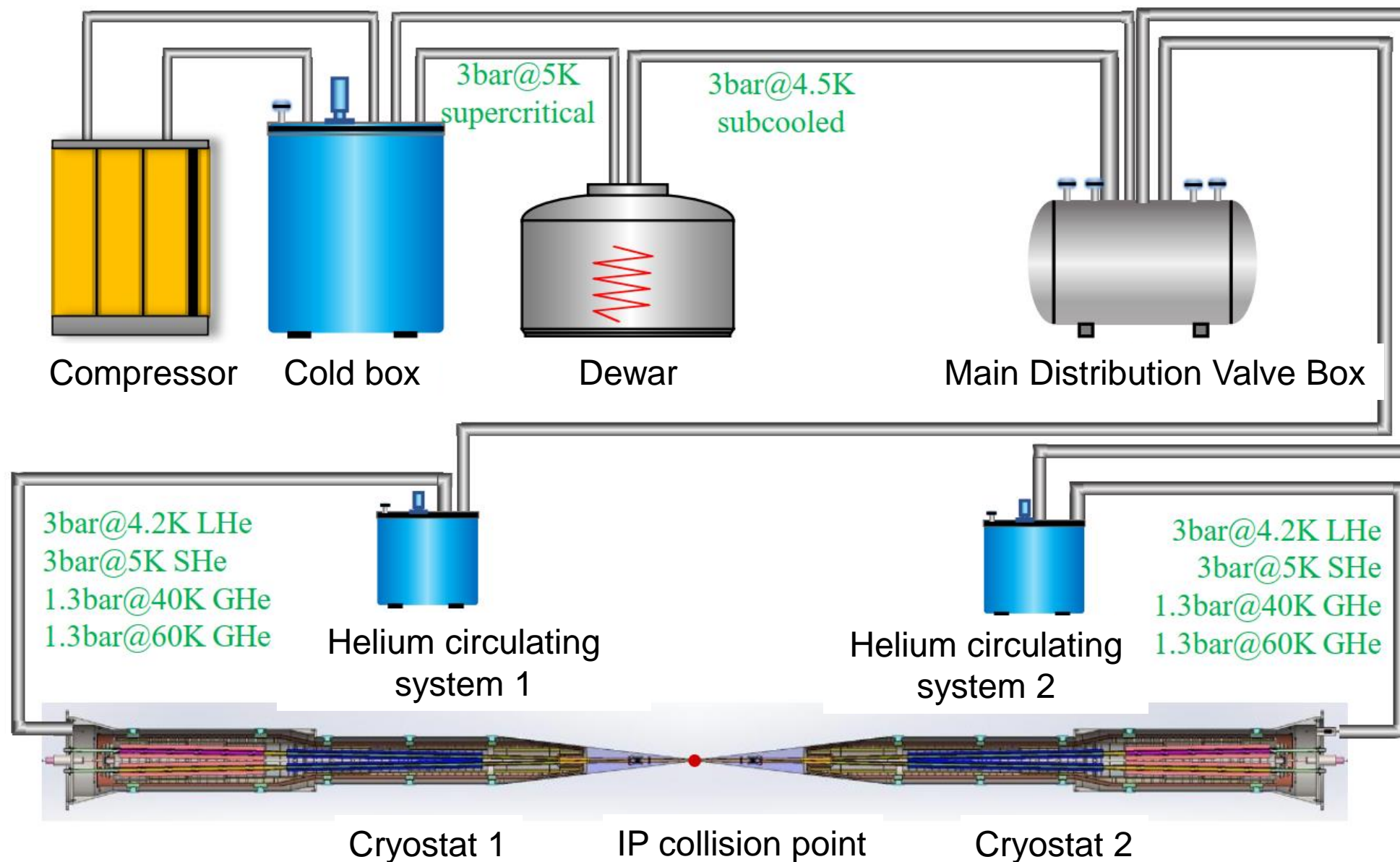
•Two individual refrigerators (2 x 4kW@4.5K) will be employed for the CEPC SC magnets cryogenic system.

- ◆ The requirements of the Final Focus quadrupoles(QD0 and QF1) are based on the length of 2.2 m, beam crossing angle of 33 mrad in the interaction region

ITEM	QD0	QF1	Anti-Solenoids
Central field gradient (T/m)	136	110	
Cooling type	LHe	LHe	LHe
Operating current (A)	2080	2300	1000
Magnetic storage energy (KJ)	25	30.5	715
Operating temperature (K)	4.5K	4.5K	4.5K
Magnet length (m)	2	1.48	1.1+2+1.7
X direction Lorentz force/octant(KN)	68	110	?
Y direction Lorentz force/octant(KN)	-140	-120	?

The total weight of magnet is about 1T, the total energy release to cryogenic system when magnet quench is about 400KJ in 2 second

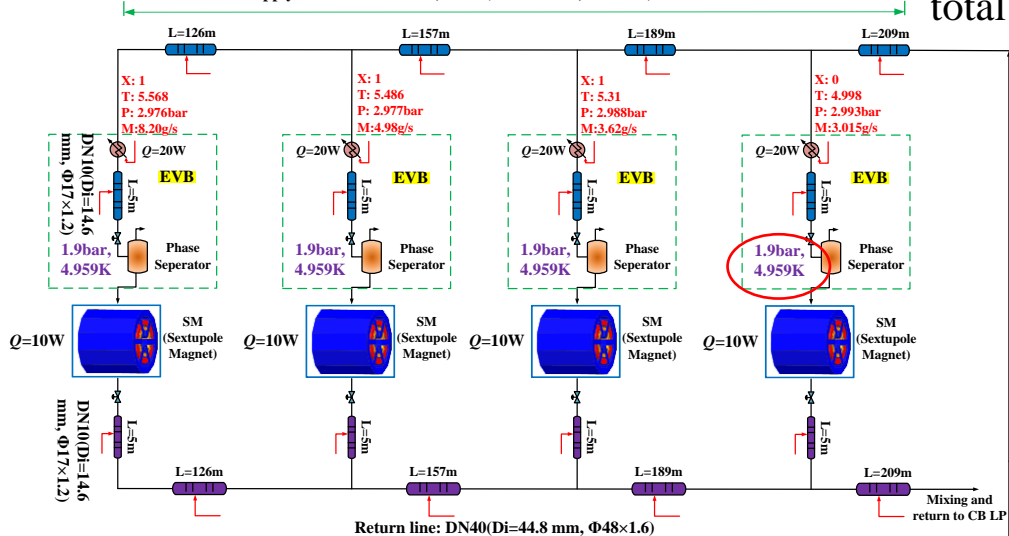
Detailed Flow Chart for CEPC MDI Cryostat (one site)



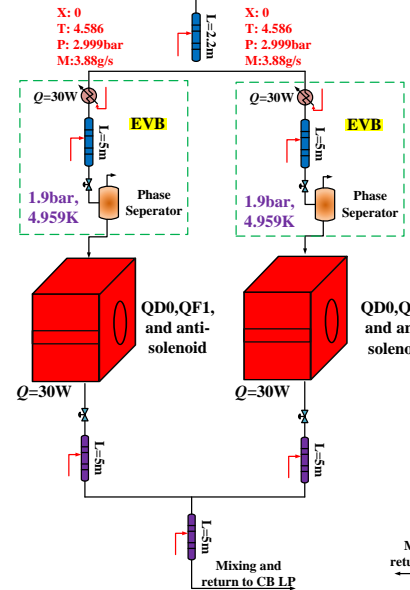
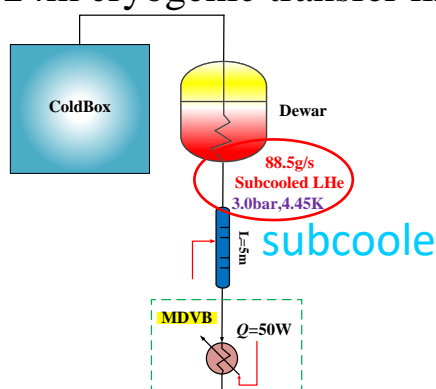
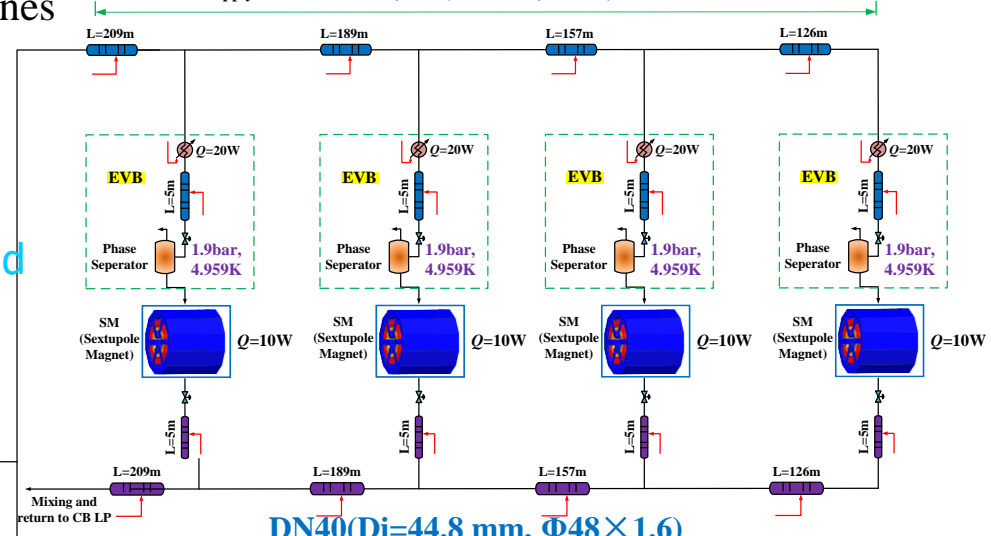
0.15W/m single loop transfer line heat loss
total 3424m cryogenic transfer lines

DN25(Di=30.8 mm, $\Phi 34 \times 1.6$)

Supply transfer line: 681m, DN25(Di=30.8 mm, $\Phi 34 \times 1.6$)



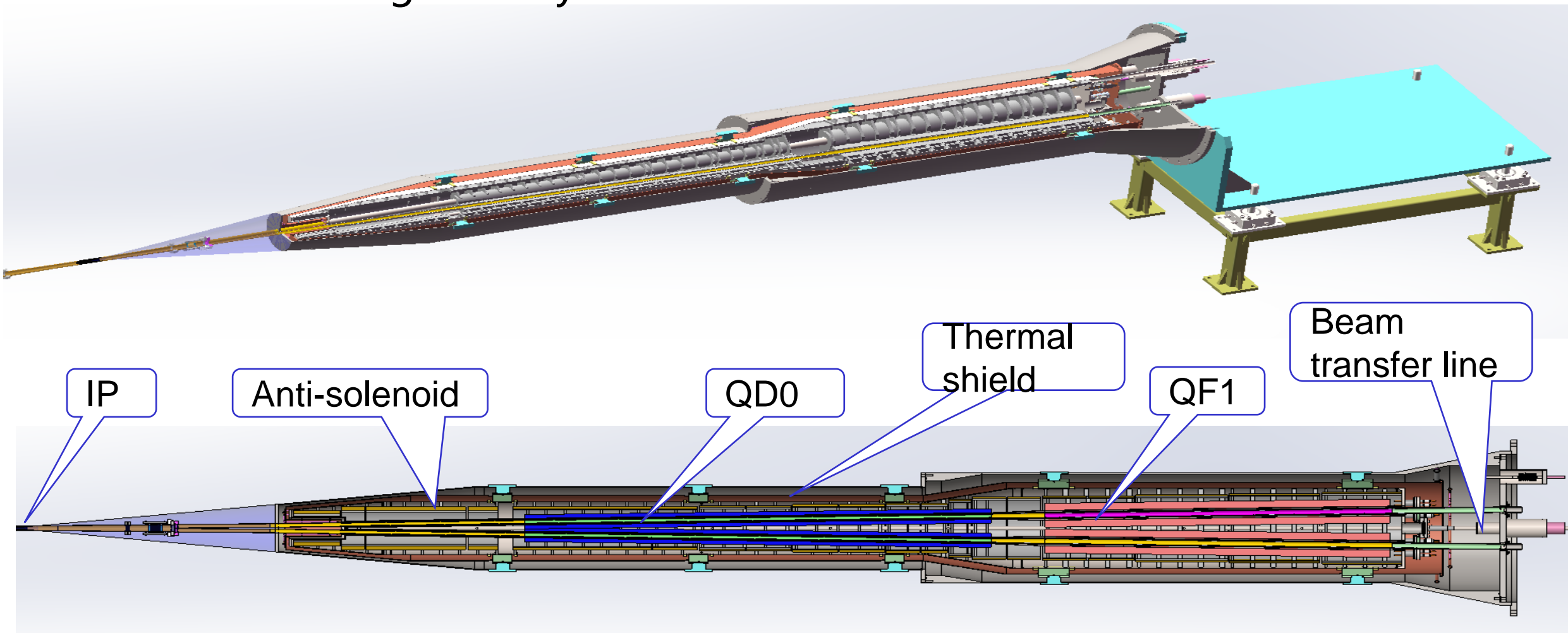
Supply transfer line: 681m, DN25(Di=30.8 mm, $\Phi 34 \times 1.6$)



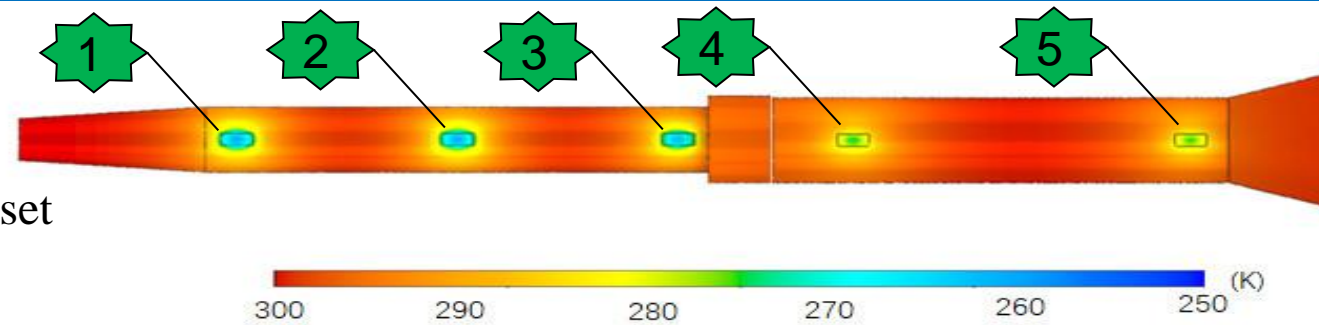
◆ Cooling method: subcooled supercritical helium.

◆ 3.0 bar@4.45K subcooled liquid helium is transferred, throttling at local equipment valve box, then return to the dewar.

- The total weight of cryostat is about 2.5 T

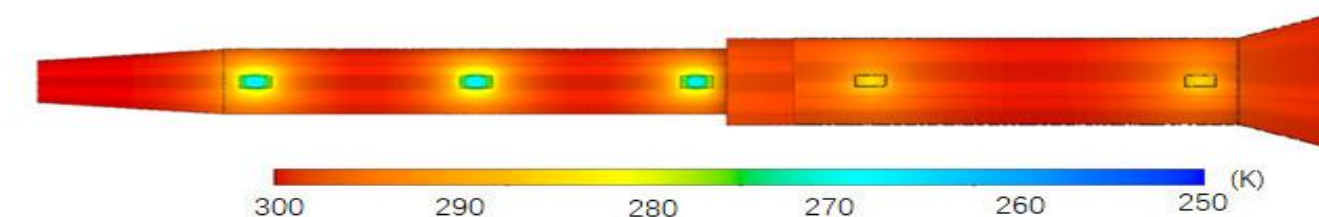


5 [$\text{W m}^{-2} \text{K}^{-1}$]

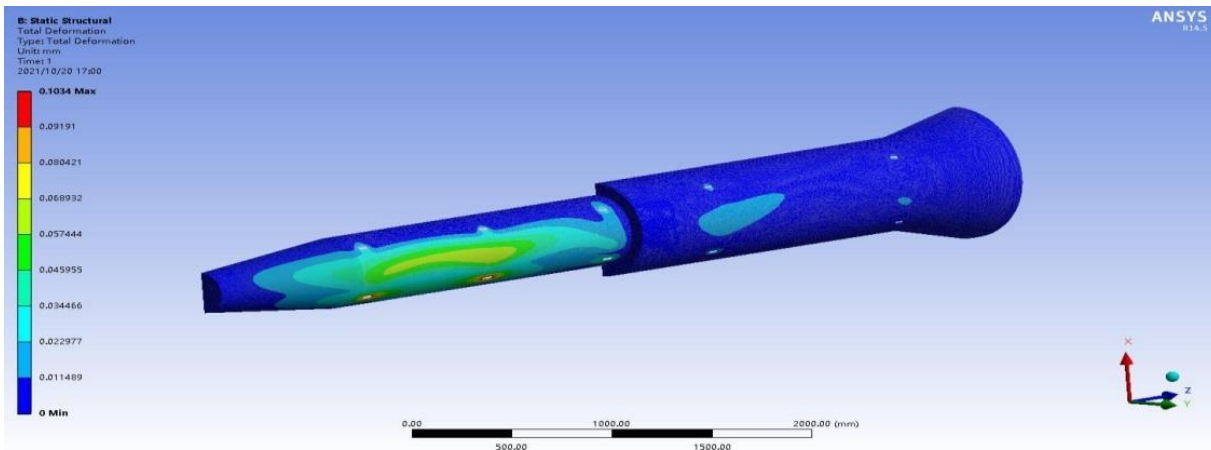


Calculation conditions: the outermost layer is set as convective boundary condition, the surface that the support structure contacts with the helium pool is set as 5K, and the surface that contacts with the cold screen is set as 60K

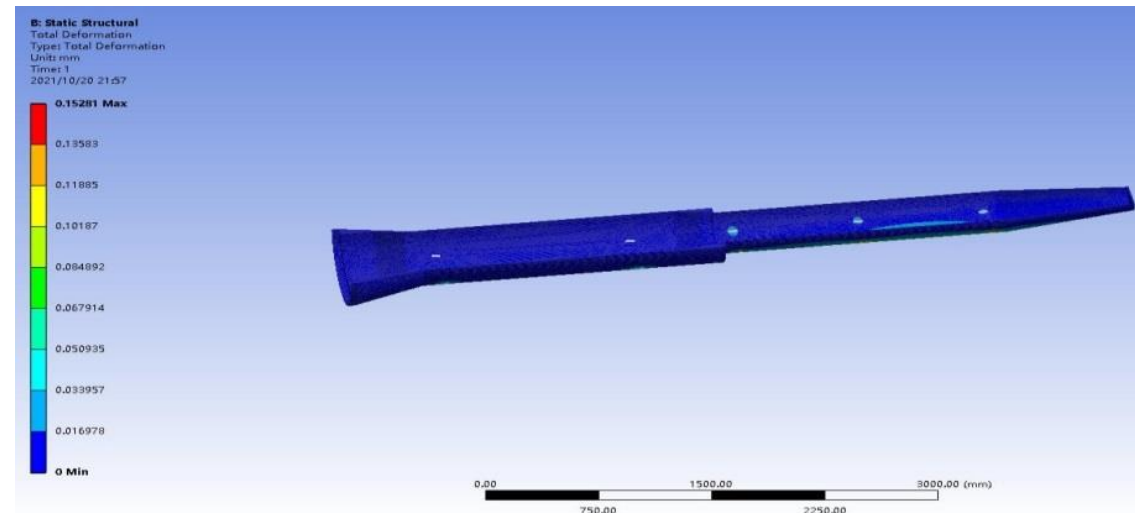
8 [$\text{W m}^{-2} \text{K}^{-1}$]



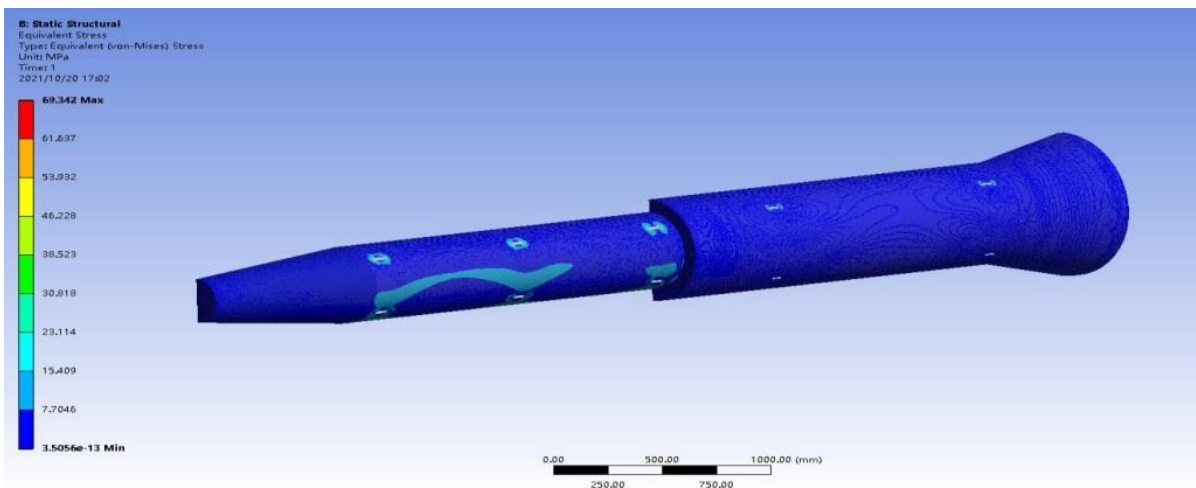
Support numbers					
convective heat transfer coefficient	1	2	3	4	5
5W $\text{m}^{-2} \text{K}^{-1}$	268.97 K	268.98 K	270.52 K	282.76 K	283.32 K
8W $\text{m}^{-2} \text{K}^{-1}$	273.14 K	273.14 K	274.09 K	285.34 K	286.65 K



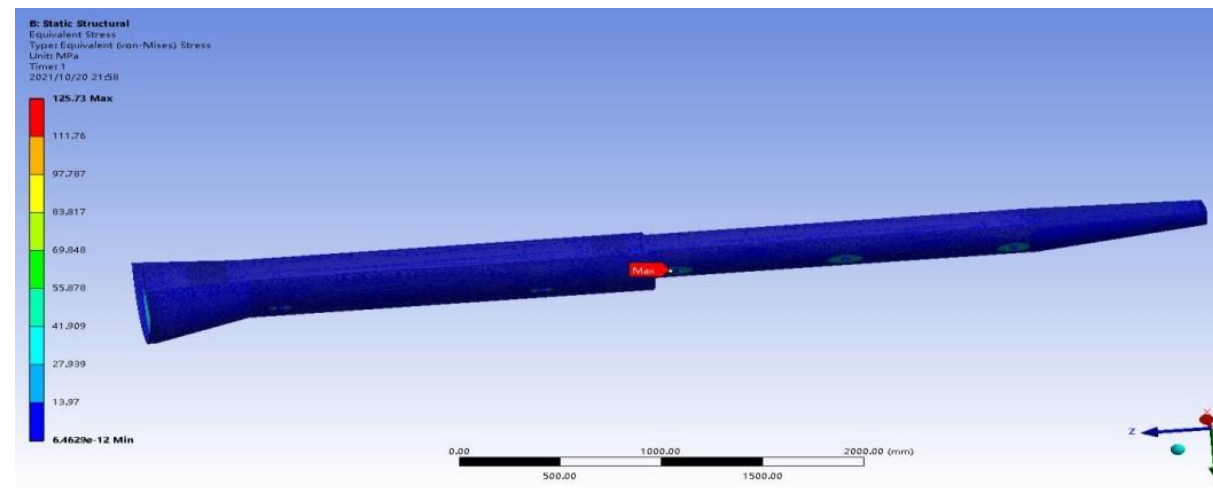
Maximum deformation 0.1mm@external pressure 1Bar



Maximum deformation 0.15mm@internal pressure 1.2Bar



Maximum stress 69.343MPa@external pressure 1Bar



Maximum stress 125.73MPa@ @internal pressure 1.2Bar

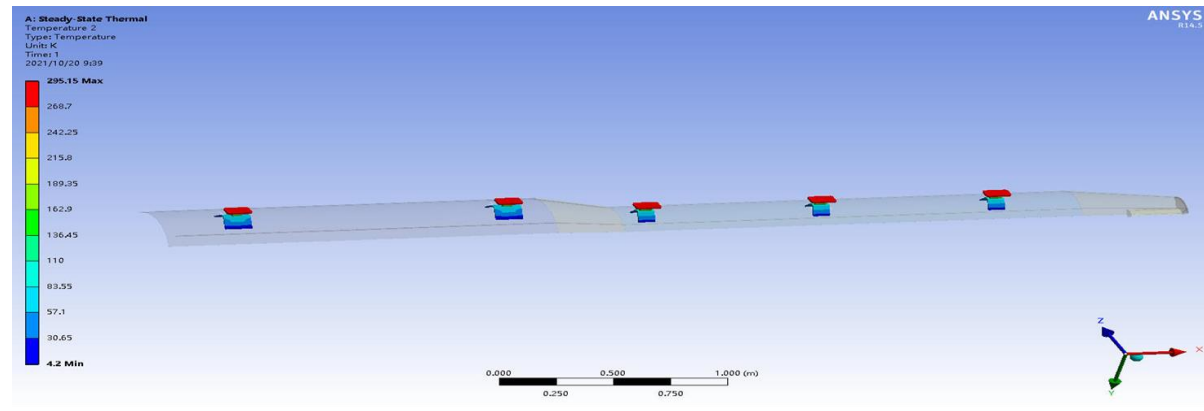
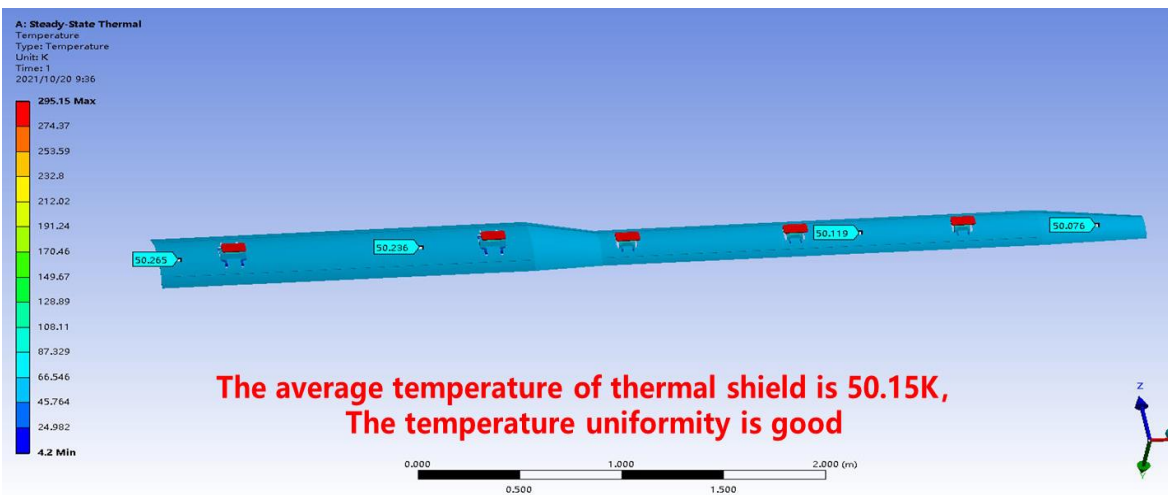


Thermal shield and supports--temperature field



Thermal shield: inlet 40K cold helium; outlet 70K cold helium. Set the temperature of cold screen coil is 50K

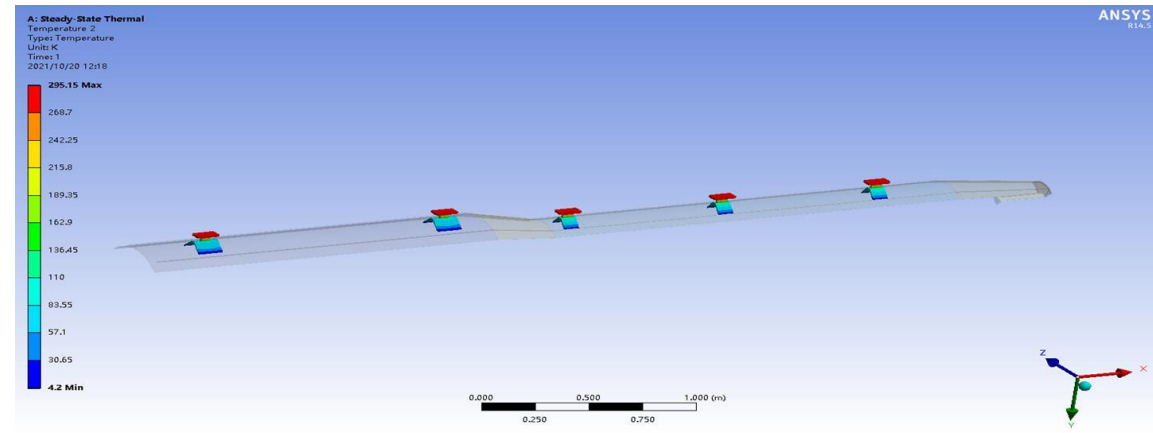
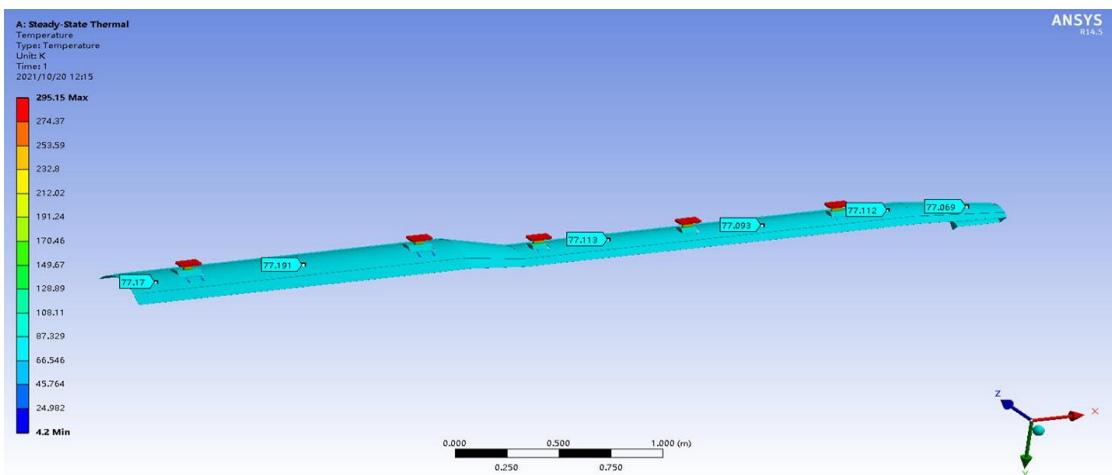
Boundary conditions: $T_{INLET}=40K$ cold helium gas, $T_{OUTLET}=70K$; Set thermal shield coils temperature is 50K.



Heat load of thermal shield: $7.12 \times 4 = 28.48W@50K$
Heat load for 4.2K temperature zone: $2.01 \times 4 = 8.04W@4.2K$

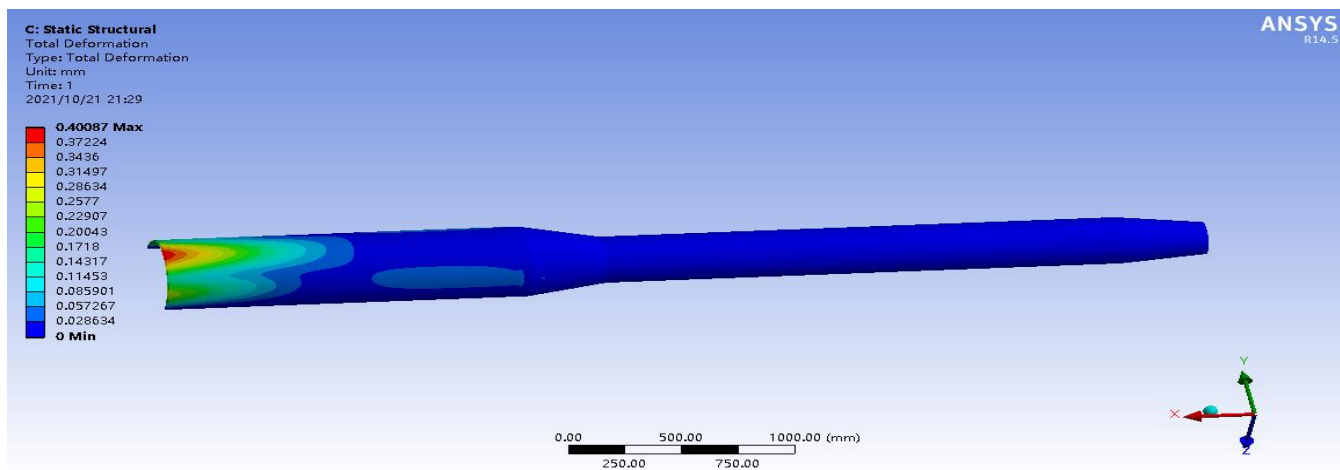
Thermal shield cooled by liquid nitrogen, Set thermal shield coils temperature is 77K

Thermal shield cooled by liquid nitrogen, Set thermal shield coils temperature is 77K

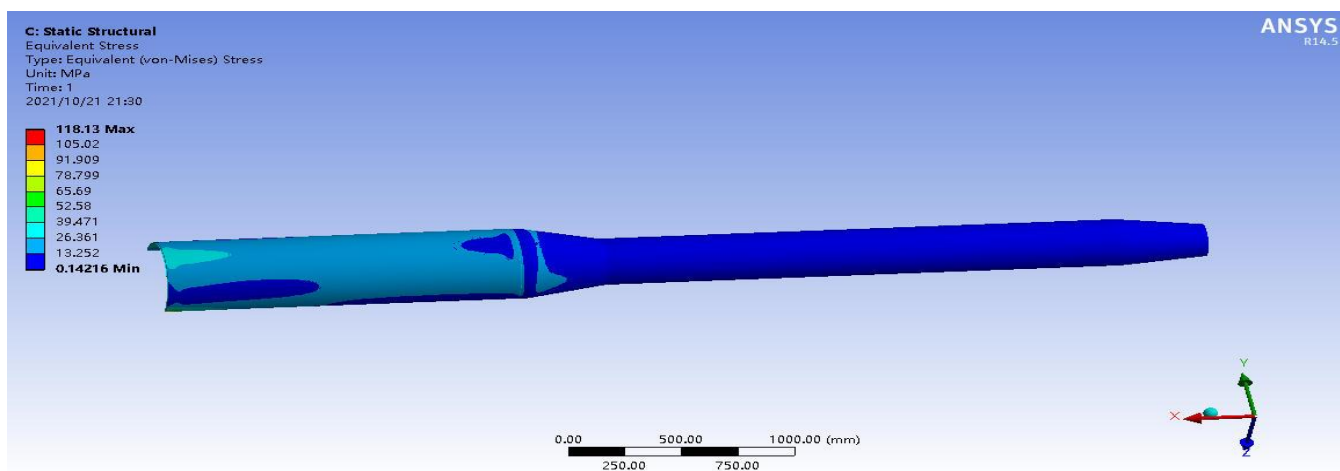


Heat load of thermal shield: $5.35 \times 4 = 21.4W@77K$
Heat load for 4.2K temperature zone: $3.42 \times 4 = 13.68W@4.2K$

Material:316L, $E=1.93E11$ Pa, Poisson' s ratio: 0.27, $[\sigma_{allow}]=120.3$ MPa



Maximum deformation为
0.4mm@P_{IN}3Bar

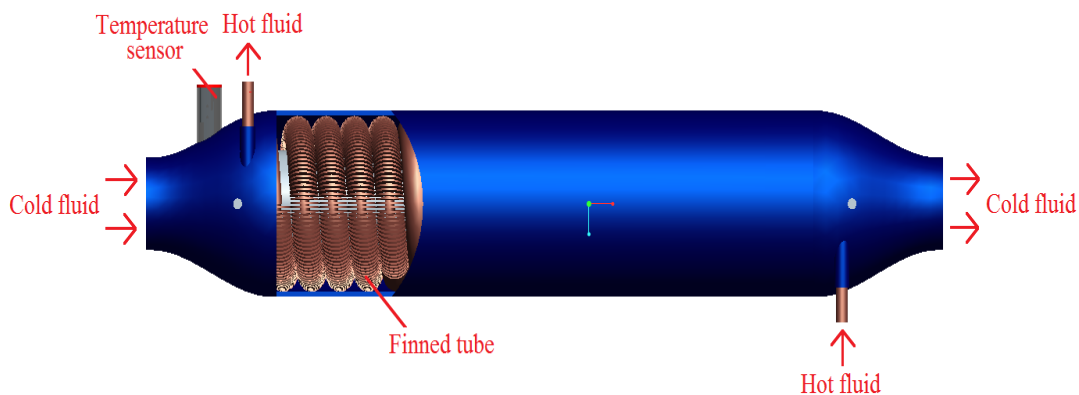


Maximum stress
118.13MPa@P_{IN}3Bar

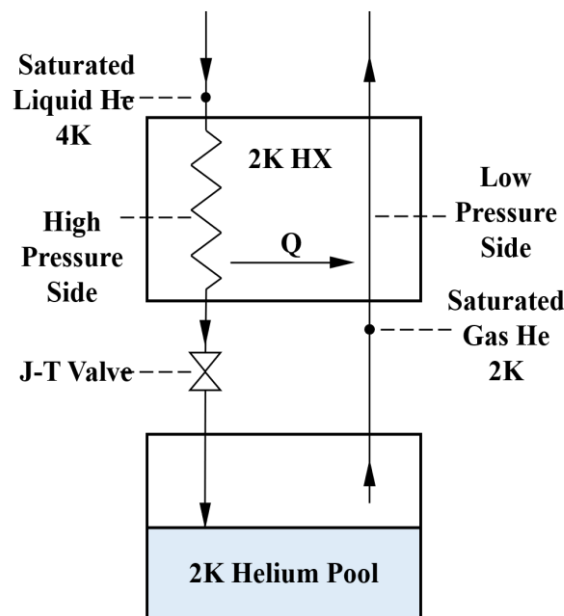
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- Summary

2K JT HX

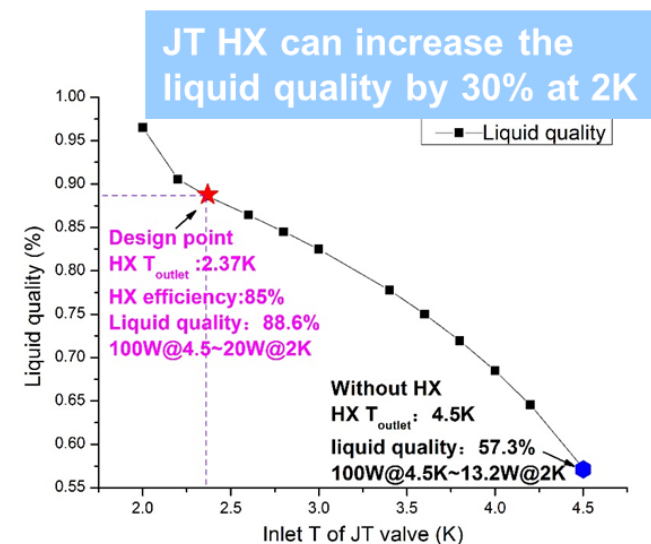
- **Function:** **key component** of the 2K cryogenic system ; Increase the liquid quality by 30% at 2K
- **Indicators:** $\varepsilon = \frac{H_{h,in} - H_{h,out}}{H_{h,in} - H_{in}^*}$ Effectiveness of HX **GHe ΔP** Pressure drop of the gas helium side



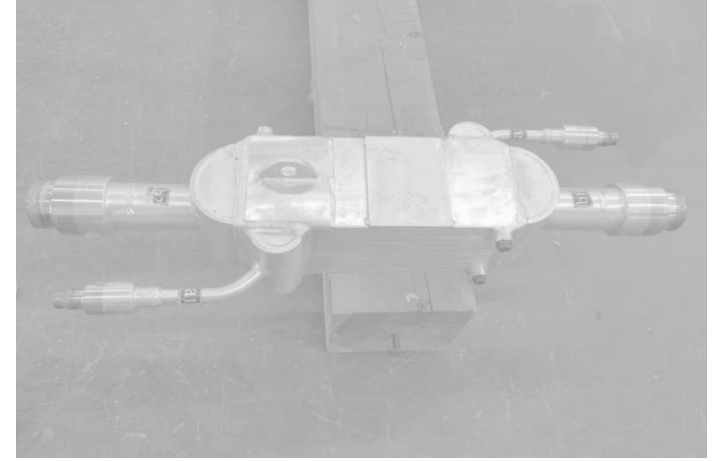
Structure of Coiled Finned-tube HX



Working Flowchart of 2K HX



Currently, 2K JT heat exchangers have been gradually developed and tested for high flow rates (10g/s → 50g/s → 130g/s (CEPC demands, kilo watts level) → >200g/s)

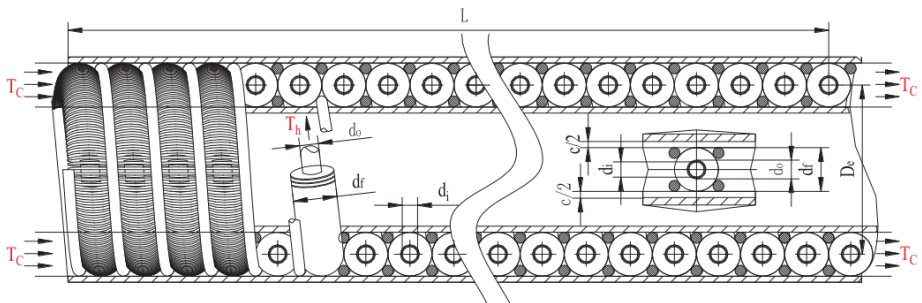


Stage1. Single Layer Coiled Finned-Tube HX

- Low mass flow rate $\leq 5\text{g/s}$
- 2K HX test Dewar's design & construction

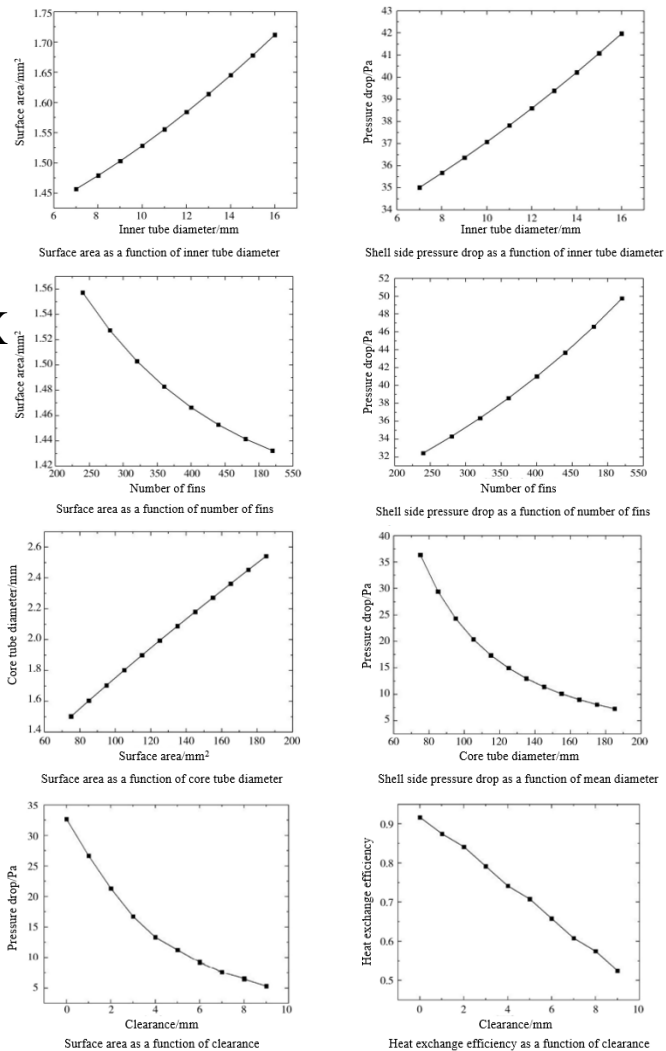
Stage1. Single Layer Coiled Finned-Tube HX

- Design



Structure of Single Layer Coiled Finned-tube HX

- Optimization



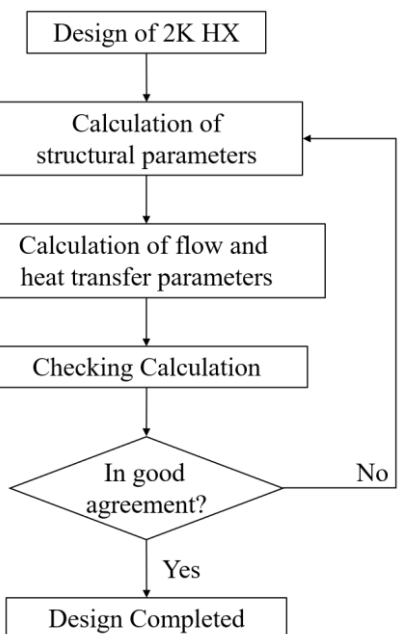
Optimal design parameters

Parameter	Value
Core tube	435mm × φ70mm
Finned-tube inner diameter	10mm
Fin height	5mm
Fin pitch	2.101mm

- Manufacture

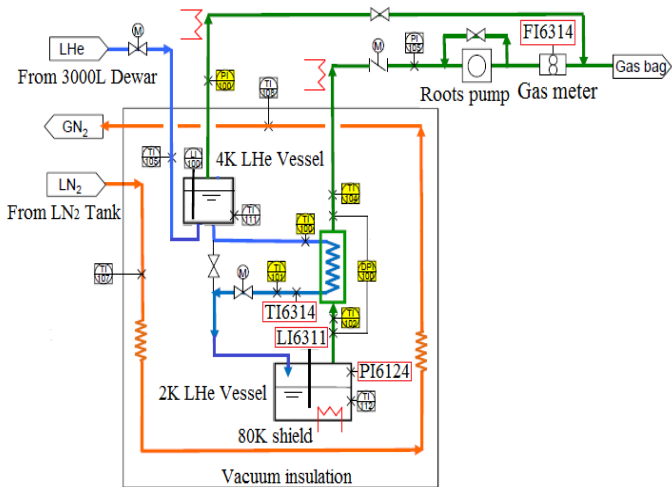


Photo of the manufactured Single Layer Coiled Finned-tube HX

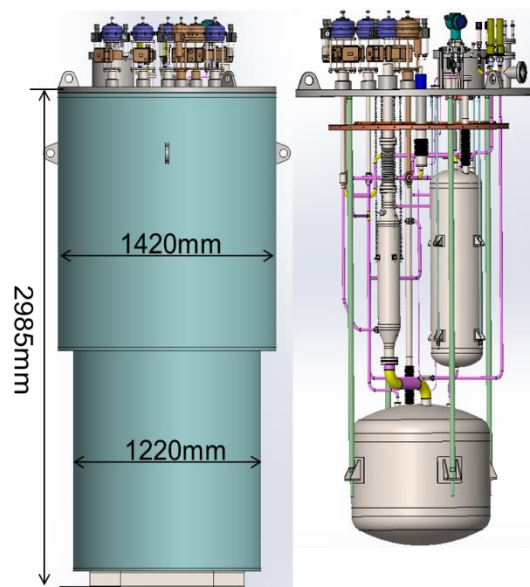


Flowchart of the design process

• Test Dewer Construction



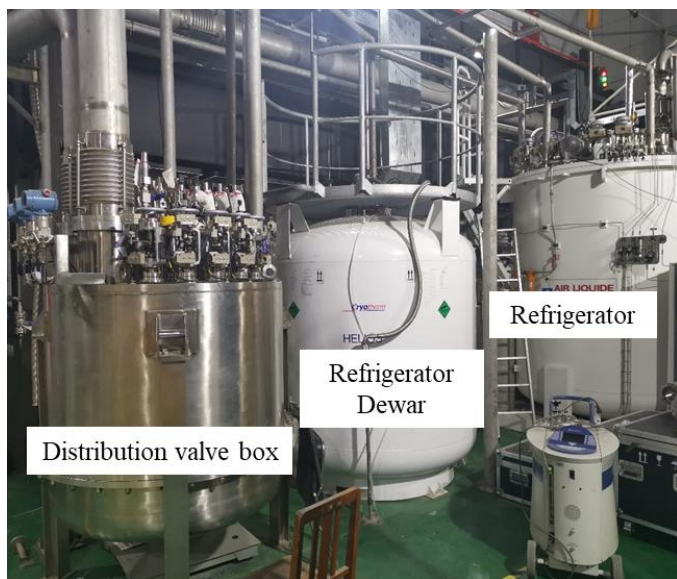
Flowchart



Test Cryostat

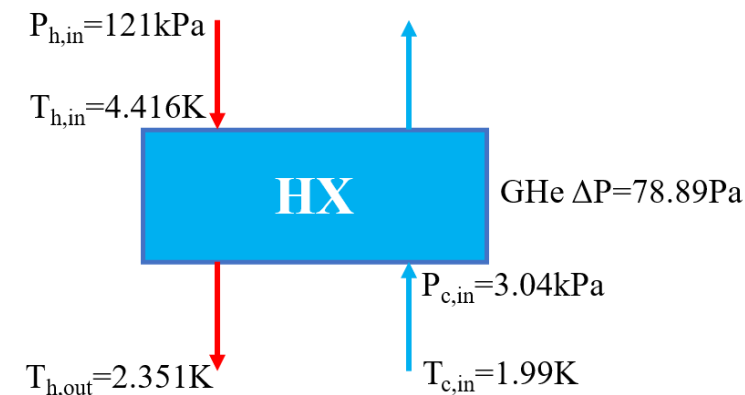


Photo of test cryostat

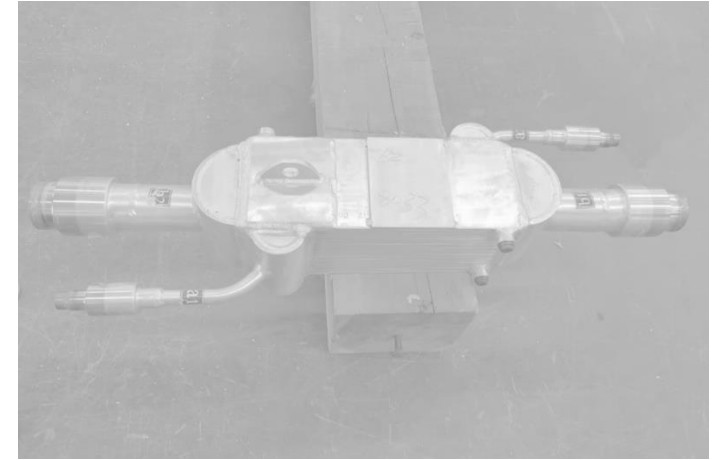


• Test Results

- design mass flow rate : **5g/s**
- Effectiveness : **80.3%**
- Pressure drop : **78.89Pa**



Test Result of the Single Layer Coiled Finned-tube HX

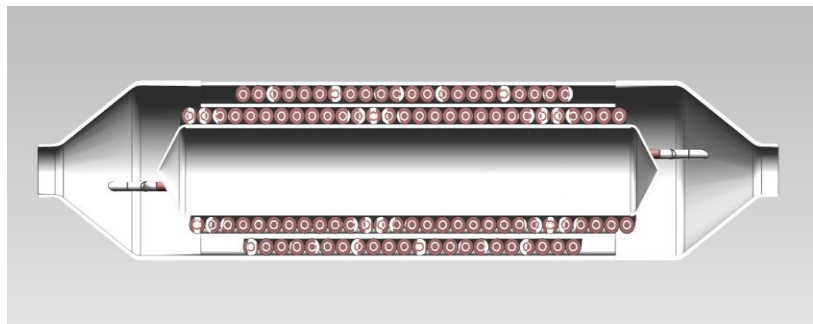


Stage2. Double Layer Coiled Finned-Tube HX

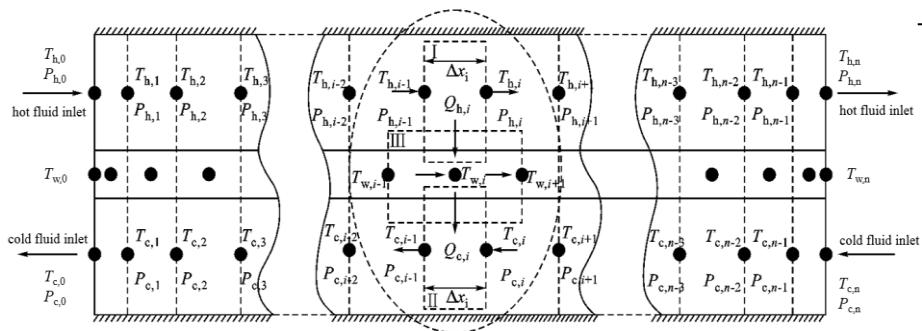
- Medium mass flow rate for 10g/s
- Distributed parameter differential method

Stage2. Double Layer Coiled Finned-Tube HX

- Design



Structure of Double Layer Coiled Finned-tube HX



Distributed parameter differential method

- Optimization

Optimal design parameters

Parameter	Value
Inner layer core tube	614mm × φ114mm
Outer layer core tube	573mm × φ169mm
Finned-tube inner diameter	8mm
Fin height	4.8mm
Fin pitch	3.125mm



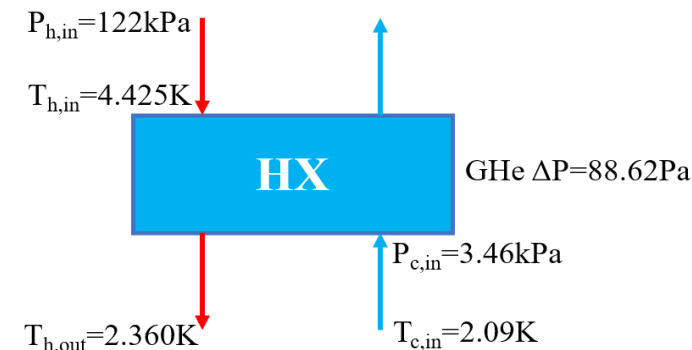
Double Layer Coiled Finned-tube HX

- Manufacture



- Test Results

- design mass flow rate : **10g/s**
- Effectiveness : **84.9%**
- Pressure drop : **88.62Pa**





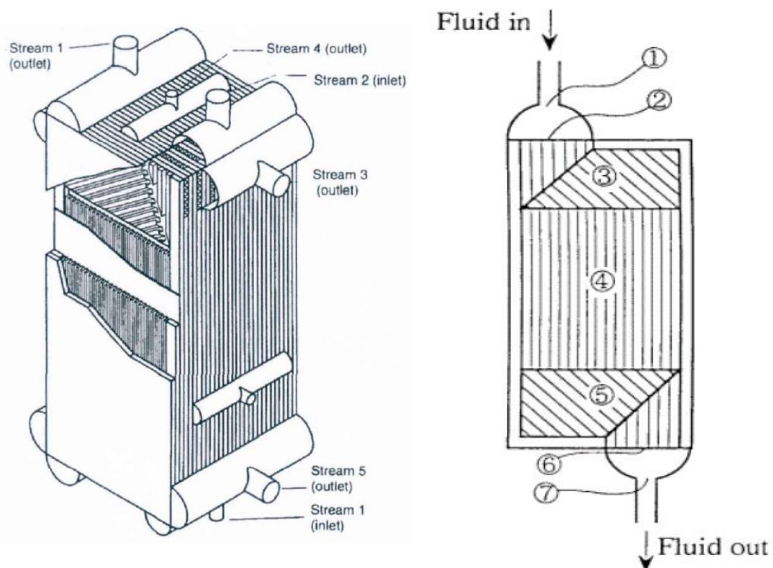
Stage3. Plate Fin HX

- Large mass flow rate for $> 10\text{g/s}$
- Multi-objective Optimization

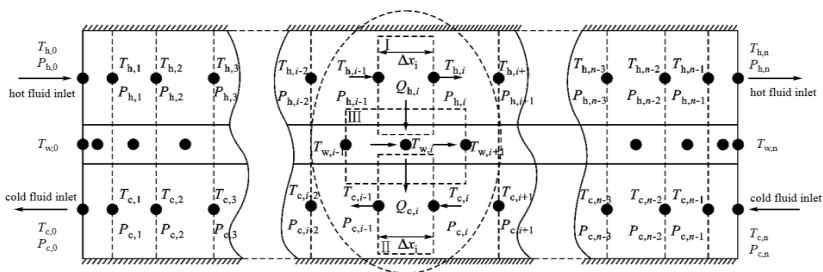
• Design Calculation

• Check Calculation

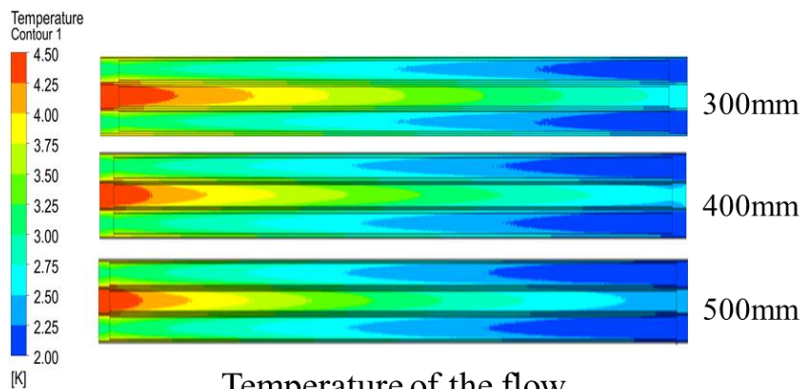
• GA Multi-objective Optimization



Structure of Plate Fin HX



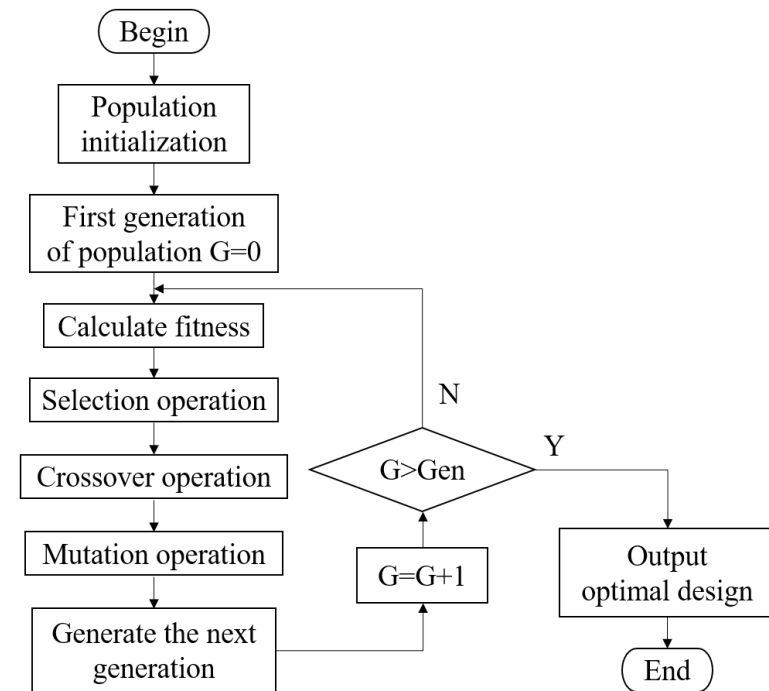
Distributed parameter differential method



Temperature of the flow

Result comparison

Length	CFD Result		Calculated Result	
	$T_{c,out}$	$T_{h,out}$	$T_{c,out}$	$T_{h,out}$
300mm	3.13K	2.95K	3.17K	3.05K
400mm	3.16K	2.76K	3.19K	2.88K
500mm	3.22K	2.58K	3.29K	2.79K



Flowchart of GA Multi-objective Optimization

$$Fit(x_1, x_2, x_3 \dots x_n) = k_1 \cdot \frac{Nu}{f^{1/2}} - k_2 \cdot V - k_3 \cdot Rt - k_4 \cdot Pc + k_5 \cdot \varepsilon$$

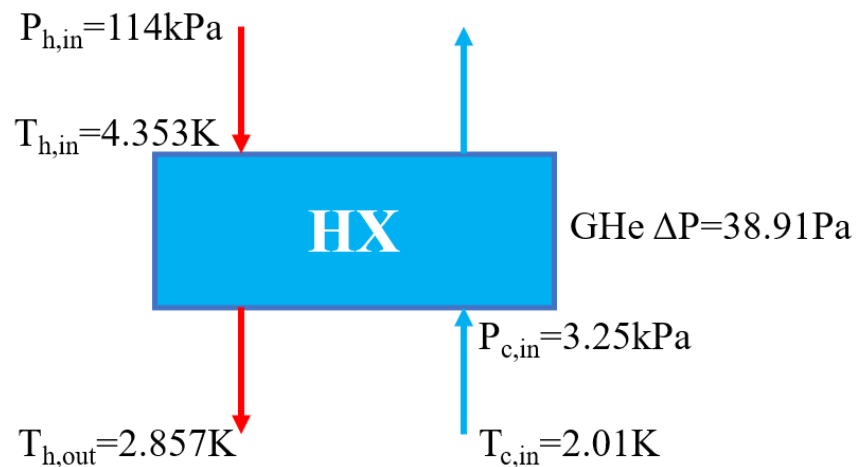
• Manufacture



Manufacture process of Plate Fin HX

• Test Results

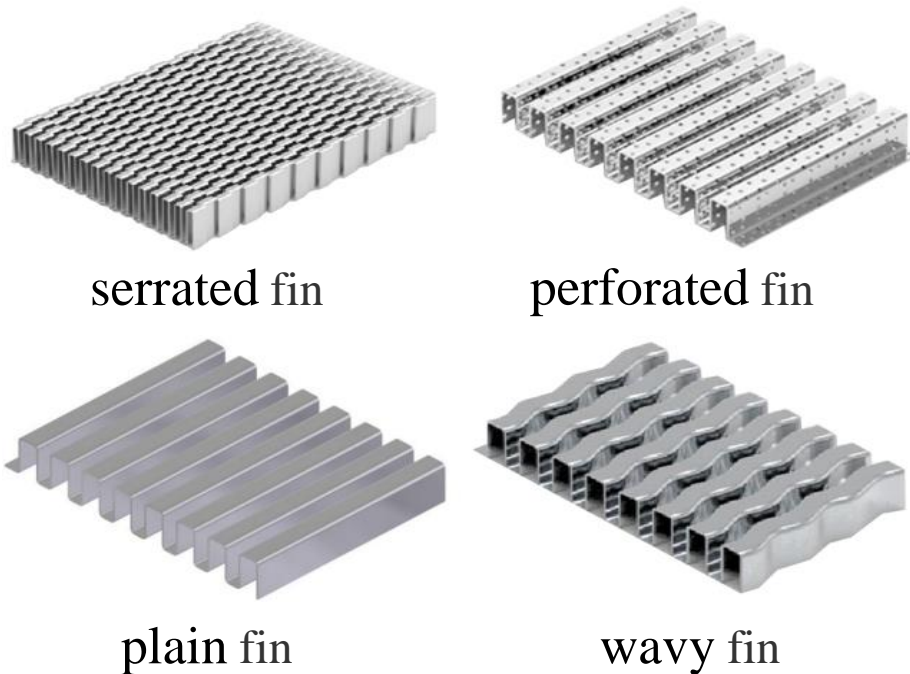
- design mass flow rate : **10g/s**
- Effectiveness : **65.2%**
- Pressure drop : **38.91Pa**



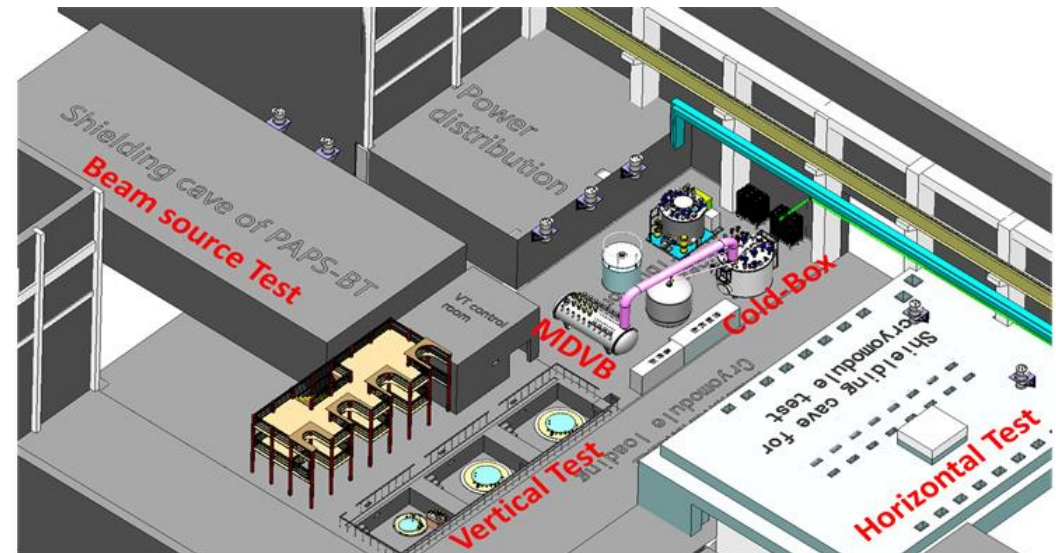
Test Result of the Single Layer Coiled Finned-tube HX

Upcoming research work

- **Research of various types of fins**
- Fins with **different shapes and structures** will be studied in order to realize the simulation of thermal and pressure drop performance of PFHE with different fin designs.



- **Construction of test Dewar for plate fin heat exchanger (PFHE) in kilowatt scale**
- Based on the cryogenic system of PAPS, the **2K heat exchanger test Dewar** will be designed and constructed to meet the experimental requirements of PFHE in **kilowatt scale**.



- ◆ Previous CDR and TDR works have laid a good foundation for the further design.
- ◆ For the CEPC cryogenic system TDR design, relevant aspects have been considered , including process scheme, layout, and key equipment design.
- ◆ We still need to work together to greatly push forward the work, and complete the TDR design on time.



Thanks for Your Attention !